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DIGITAL COMPUTER SIMULATION FOR SURFACE
SHIP CONTROL

Aporm Ratanaruang

Naval Postgraduate School
Monterey, California

June 1973

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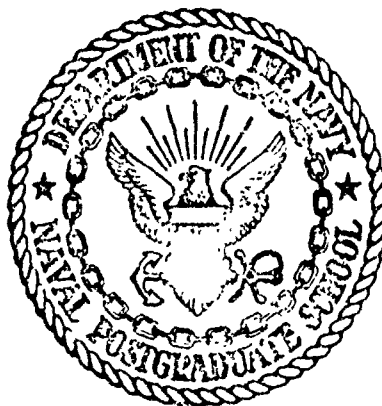
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Monterey, California



THESIS

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Aporn Ratanaruang

Thesis Advisor:

M. L. Wilcox

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Digital Computer Simulation
for
Surface Ship Control

by

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The general equations of surface ship motion are developed and standardized for simulation in digital computer. Digital simulations of the dynamics of the surface ship in three degrees of freedom are done with and without non-linear and cross-coupling terms.

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I. EQUATIONS OF SURFACE SHIP MOTION

A moving ship is a body with six degrees of freedom. These degrees of freedom are generally chosen as follows:

a. Linear displacements along the three axes through the center of gravity.

a.1 Surge -- along X axes

a.2 Sway -- along Y axes

a.3 Heave -- along Z axes

b. Rotations around the three axes through the center of gravity.

b.1 Roll -- around X axes

b.2 Pitch -- around Y axes

b.3 Yaw -- around Z axes

Further reduction in the complex nature of the equations can be brought about by choosing an orthogonal axis system parallel to the principal axes of inertia so as to eliminate products of inertia in the motion equations. For practically all ocean vehicles, with extremely few exceptions, a longitudinal axis (X axis) in the centerline plane, a downward (toward keel) axis (Z axis) perpendicular to the X axis in the centerline plane, and a transverse axis (Y axis) perpendicular to the centerline plane satisfies this requirement.

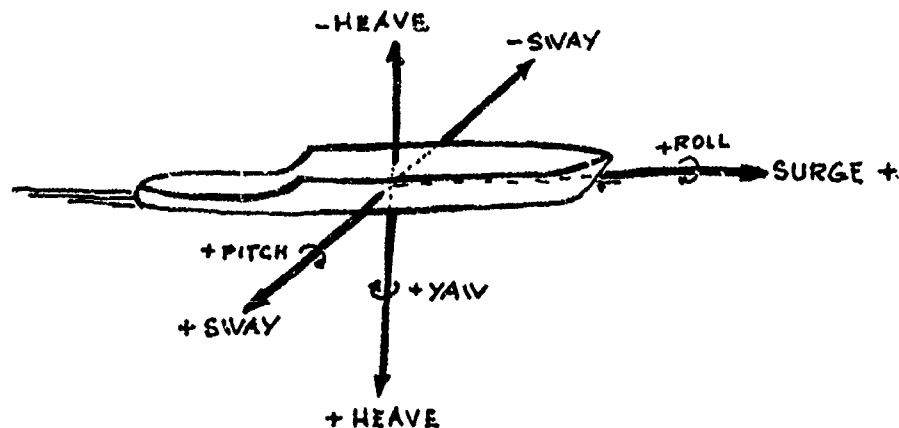


Figure 1. Surface ship in six degrees of freedom.

For the exceptional vehicle which has a very peculiar and significantly large asymmetrical mass distribution, it is necessary to include the products of inertia.

The X, Y, Z axes form an orthogonal right hand system of axes fixed in the vehicle. The axes and the associated components of the pertinent physical quantities are defined below:

The longitudinal X axis (in the plane of symmetry) is positive in the forward direction, usually parallel to the keel or calm water line. If the upper and lower halves of the body are symmetrical, then the axis is the intersection of the two planes of symmetry.

The Y axis is the transverse axis perpendicular to the plane of symmetry and positive to the starboard.

The Z axis or downward axis in the plane of symmetry (X,Z) is perpendicular to the X axis and positive downward towards the keel.

$\hat{i}, \hat{j}, \hat{k}$ unit vectors along the X, Y, Z axis respectively.

\vec{R} x, y, z vector distance of a point from the origin O, and the corresponding components along the X, Y and Z axes.

$$\vec{R} = ix_G + jy_G + kz_G$$

\vec{U} u, v, w velocity of the origin O (on the body) and the corresponding components along the X, Y and Z axis.

$$\vec{U} = \hat{i}u + \hat{j}v + \hat{k}w$$

$\vec{\Omega}$ p, q, r angular velocity of the body about the origin and the corresponding components about the axes.

$$\vec{\Omega} = \hat{i}p + \hat{j}q + \hat{k}r$$

The moments of inertia of the body about the X, Y and Z axes respectively I_x, I_y, I_z .

\vec{F} X, Y, Z, force acting on the body and the corresponding components along the axes.

$$\vec{F} = \hat{i}X + \hat{j}Y + \hat{k}Z$$

\vec{M} K, M, N Moments acting about the axes.

$$\vec{M} = \hat{i}K + \hat{j}M + \hat{k}N$$

Newton's law of motion for a rigid body can be written as two equations, one a force equation and the second a moment equation provided an origin is taken at the center of gravity and the axis system is fixed in space. The equations are

$$\vec{F} = \frac{d}{dt} (\overrightarrow{\text{momentum}}) = \frac{d}{dt} (m \vec{U}_G)$$

$$\vec{M} = \frac{d}{dt} (\overrightarrow{\text{angular momentum}})_G = \frac{d}{dt} (I \vec{\Omega})$$

where the subscript G refers to an origin at the center of gravity and m is the mass of the body. For a mass essentially constant in time

$$\vec{F} = m \frac{d}{dt}(\vec{U}_G)$$

For an origin not at the center of gravity of the body and in a system of axes fixed in and moving with the vehicle.

$$\vec{U}_G = U_a + \vec{\Omega} \times \vec{R}_G$$

where U_a is the velocity of the origin in space. However, since the origin is on the surface of the earth and the earth rotates, then

$$\vec{U}_a = \vec{U} + \vec{\Omega}_e \times \vec{R}_b$$

where \vec{U} is the geographical velocity of the body, $\vec{\Omega}_e$ is the angular velocity of the earth, and \vec{R}_b is the radius vector from earth's center to the vehicle. The force equation becomes:

$$\vec{F} = m \frac{d}{dt}(\vec{U} + \vec{\Omega}_e \times \vec{R}_b + \vec{\Omega} \times \vec{R}_G)$$

$$\begin{aligned} \vec{F} &= m \frac{d}{dt}(\vec{U} + \vec{\Omega} \times \vec{R}_b) + m[\dot{\vec{\Omega}}_e \times \vec{R}_b + \vec{\Omega}_e \times \dot{\vec{R}}_b] \\ &= m \frac{d}{dt}(\vec{U} + \vec{\Omega} \times \vec{R}_G) + m[\dot{\vec{\Omega}}_e \times \vec{U} + \vec{\Omega}_e \times \dot{\vec{\Omega}}_e \times \vec{R}_b] \end{aligned}$$

since $\dot{\vec{\Omega}} = 0$ and $\dot{\vec{R}}_b = \vec{U} + \vec{\Omega}_e \times \vec{R}_b$

the term $m\dot{\vec{\Omega}}_e \times \vec{U}$ is the coriolis force and $m\vec{\Omega}_e \times \dot{\vec{\Omega}}_e \times \vec{R}_b$ is the centripetal acceleration due to rotation of the

earth. These two terms are negligibly small when compared with the other forces, then

$$\vec{F} = m \frac{d}{dt}(\vec{U} + \vec{\Omega} \times R_G)$$

Finding the derivatives of unit vectors (change in direction)

$$\begin{array}{lll} \text{where} & \hat{i} = -\hat{k} d\theta & \hat{i} = \hat{j} d\psi & \hat{i} = 0 \\ & \hat{j} = 0 & \hat{j} = -\hat{i} d\psi & \hat{j} = \hat{k} d\phi \\ & \hat{k} = \hat{i} d\theta & \hat{k} = 0 & \hat{k} = -\hat{j} d\phi \end{array}$$

Adding the contributions

$$\frac{d\hat{i}}{dt} = \hat{i} \cdot 0 + \hat{j} \frac{d\psi}{dt} - \hat{k} \frac{d\theta}{dt}$$

$$\frac{d\hat{j}}{dt} = -\hat{i} \frac{d\psi}{dt} + \hat{j} \cdot 0 + \hat{k} \frac{d\phi}{dt}$$

$$\frac{d\hat{k}}{dt} = \hat{i} \frac{d\theta}{dt} - \hat{j} \frac{d\phi}{dt} + \hat{k} \cdot 0$$

$$\vec{\Omega} = \hat{i}p + \hat{j}q + \hat{k}r$$

$$p = \frac{d\phi}{dt}, \quad q = \frac{d\theta}{dt}, \quad r = \frac{d\psi}{dt}$$

$$\frac{d\hat{i}}{dt} = \vec{\Omega} \times \hat{i} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ p & q & r \\ 1 & 0 & 0 \end{vmatrix}$$

$$\vec{U} = \hat{i}u + \hat{j}v + \hat{k}w$$

$$\vec{R}_G = \hat{i}x_G + \hat{j}y_G + \hat{k}z_G$$

$$\frac{d\vec{U}}{dt} = \hat{i}\dot{u} + u(\hat{j}\frac{d\hat{i}}{dt} + \hat{j}\dot{v} + v\frac{d\hat{j}}{dt} + \hat{k}\dot{w} + w\frac{d\hat{k}}{dt})$$

$$= \hat{i}\dot{u} + u(\hat{j}\frac{d\psi}{dt} - \hat{k}\frac{d\theta}{dt}) + \hat{j}\dot{v} + v(-\hat{i}\frac{d\psi}{dt} + \hat{k}\frac{d\phi}{dt})$$

$$+ \hat{k}\dot{w} + w(\hat{i}\frac{d\theta}{dt} - \hat{j}\frac{d\phi}{dt})$$

$$= \hat{i}\dot{u} + u(\hat{j}r - \hat{k}q) + \hat{j}\dot{v} + v(-\hat{i}r + \hat{k}p) + \hat{k}\dot{w} + w(\hat{i}q - \hat{j}p)$$

$$\vec{\Omega} \times \vec{R}_G = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ p & q & r \\ x_G & y_G & z_G \end{vmatrix} = \hat{i}(qz_G - ry_G) - \hat{j}(pz_G - rx_G) + \hat{k}(py_G - qx_G)$$

$$\begin{aligned} \frac{d}{dt}(\vec{\Omega} \times \vec{R}_G) &= \hat{i}(z_G \dot{q} - y_G \dot{r}) + \hat{j}(z_G \dot{q} r - y_G r^2) + \hat{k}(y_G \dot{q} r - z_G r^2) \\ &\quad + \hat{j}(x_G \dot{r} - z_G \dot{p}) + \hat{k}(x_G \dot{r} p - z_G p^2) + \hat{i}(z_G \dot{r} p - x_G p^2) \\ &\quad + \hat{k}(y_G \dot{p} - x_G \dot{q}) + \hat{i}(y_G \dot{p} q - x_G q^2) + \hat{j}(x_G \dot{p} q - y_G p^2) \end{aligned}$$

$$\text{from } \vec{F} = \hat{i}X + \hat{j}Y + \hat{k}Z \quad \text{and} \quad \vec{F} = m \left[\frac{d\vec{U}}{dt} + \frac{d}{dt}(\vec{\Omega} \times \vec{R}_G) \right]$$

Rewriting and grouping all \hat{i} terms equal to X, all \hat{j} terms equal to Y and all \hat{k} terms equal to Z yield

$$\begin{aligned} X &= m(\dot{u} - vr + wg + z_G \dot{q} + rpz_G - r^2 x_G + pqy_G - q^2 x_G) \\ &= m(\dot{u} + wg - vr - x_G(r^2 + q^2) + y_G(pq - \dot{r}) + z_G(rp + \dot{q})) \\ Y &= m(\dot{v} + ur - wp + x_G(pq + \dot{r}) - y_G(p^2 + r^2) + z_G(pq - \dot{r})) \\ Z &= m(\dot{w} + vp - uq + x_G(rp - \dot{q}) + y_G(qr + \dot{p}) - z_G(p^2 + q^2)) \end{aligned} \quad (1)$$

$$\begin{aligned} \text{From } \vec{M}_G &= \frac{d}{dt}(\text{angular momentum})_G \\ &= \frac{d}{dt}(\hat{i}I_{x_G}p + \hat{j}I_{y_G}q + \hat{k}I_{z_G}r) \end{aligned}$$

G indicates an origin at the center of gravity

$$I_{x_G} = I_X - m(y_G^2 + z_G^2)$$

$$I_{y_G} = I_Y - m(z_G^2 + x_G^2)$$

$$I_{z_G} = I_Z - m(x_G^2 + y_G^2)$$

$$\vec{M} = \vec{M}_G + \vec{R}_G \times \vec{F} \quad \text{or} \quad \vec{M}_G = \vec{M} - \vec{R}_G \times \vec{F}$$

After manipulating and using the results for the derivatives of unit vectors (same as above) expressions for K, H and N are obtained.

$$\begin{aligned} K &= I_X \dot{p} + (I_Z - I_Y)qr + m[Y_G(\dot{w} + pv - qu) - Z_G(\dot{v} + ru - pw)] \\ M &= I_Y \dot{q} + (I_X - I_Z)rp + m[Z_G(\dot{u} + qw - rv) - X_G(\dot{w} + pv - qu)] \quad (2) \\ N &= I_Z \dot{r} + (I_Y - I_X)pq + m[X_G(\dot{v} + ru - pw) - Y_G(\dot{u} + qw - rv)] \end{aligned}$$

The terms $(qw - rv)$, $(ru - pw)$ and $(pv - qu)$ are gyroscopic effects.

The relationship for forces and moments can be expressed

$$\begin{array}{l} \text{Force) } \\ \text{Moment) } \end{array} = f(\text{properties of body, properties of motion, properties of fluid})$$

For a particular ship, in a given fluid with no excitation force - so

$$\begin{aligned} \begin{pmatrix} \vec{F} \\ \vec{M} \end{pmatrix} &= f(\text{properties of motion}) \\ &= f(X_O, Y_O, Z_O, \phi, \theta, \psi, u, v, w, p, q, r, \dot{u}, \dot{v}, \dot{w}, \dot{p}, \dot{q}, \dot{r}, \delta, \delta', \delta'') \end{aligned}$$

The Taylor series which has the following form may now be applied to linearize the equations about an operating point \bar{X}_O

$$f(X) = f(\bar{X}_O) + (X - \bar{X}_O) \frac{d}{dx} f(\bar{X}_O) + (X - \bar{X}_O)^2 \frac{d^2}{dx^2} f(\bar{X}_O) + \dots$$

Apply this to $f(X, Y, Z) \dots \dots \dots$

For the case of f (properties of motion) let $(X - \bar{X}_0) = \Delta X_0$, $(Y - \bar{Y}_0) = \Delta Y_0$ and $(Z - \bar{Z}_0) = \Delta Z_0$

terms second order and higher are neglected for small perturbations.

From X equation the linear terms are obtained:

$$X = \bar{X}_0 + \Delta X_0 \left(\frac{\delta f}{\delta X_0} \right) + \Delta Y_0 \left(\frac{\delta f}{\delta Y_0} \right) + \Delta Z_0 \left(\frac{\delta f}{\delta Z_0} \right) + \dots \Delta v \left(\frac{\delta f}{\delta v} \right) + \dots$$

Th. defining relations are:

$$\left(\frac{\delta f}{\delta u} \right)_0 = \left(\frac{\delta X}{\delta u} \right)_{u=u_0} = X_u$$

$$\left(\frac{\delta X}{\delta w} \right)_{w=w_0} = X_w$$

$$\Delta w = (w - w_0) = w, \quad w_0 = 0$$

$$\Delta u = (u - u_0)$$

The force equations then become

$$\begin{aligned} X &= X_0 + X_{X_0} X_0 + X_{Y_0} Y_0 + \dots X_{\theta} \theta + \dots X_u \Delta u \\ Y &= Y_0 + Y_{X_0} X_0 + Y_{Y_0} Y_0 + \dots Y_{\theta} \theta + \dots Y_u \Delta u \\ Z &= Z_0 + Z_{X_0} X_0 + Z_{Y_0} Y_0 + \dots Z_{\theta} \theta + \dots Z_u \Delta u \end{aligned} \quad (3)$$

A similar derivation can be done for the K, M and N equations. The preceding X, Y and Z equations may now be equated to the linearized X, Y and Z (equations (1)), e.g., for the Y equation without roll, pitch, and the center of gravity at $X_G = 0$, $Y_G = 0$ and $Z_G = 0$ gives the linearized equation.

$$Y = m(\dot{v} + ur)$$

then

$$Y_{X_O} \dot{X}_O + Y_{Y_O} \dot{Y}_O + Y_{\psi} \dot{\psi} + Y_u \dot{u} + Y_v \dot{v} + Y_r \dot{r} + Y_{\dot{u}} \ddot{u} + Y_{\dot{v}} \ddot{v} + Y_{\dot{r}} \ddot{r} = m(\ddot{v} + ru)$$

Expressions for X, Z and K, M and N can be determined in a similar procedure.

In order to obtain the equations in a non-dimensional form some definitions will be given, and applied to the Y force equations as an example of the process.

$$\text{Froude number} = \frac{U}{\sqrt{gl}}$$

$$u', v', w' = \frac{u, v, w}{\sqrt{g/l}}$$

$$t' = t(\sqrt{g/l})$$

$$X', Y', Z' = \frac{X, Y, Z}{\rho g l^3}$$

$$C_X', C_Y', C_Z' = \frac{X, Y, Z}{\frac{1}{2} \rho U^2 l^2}$$

After replacing and adding the effect of waves, the Y equation becomes:

$$\begin{aligned} \dot{v}' + r' u' &= \frac{1}{2} U^2 (C_{Y_v'} v' + C_{Y_p'} p' + C_{Y_r'} r' + C_{Y_{\delta r}} \delta r) \\ &+ (Y_{\dot{p}} \dot{p}' + Y_{\dot{v}} \dot{v}' + Y_{\dot{r}} \dot{r}' + Y_{\dot{w}}) \end{aligned}$$

II. DIGITAL COMPUTER SIMULATION

The six equations of motion after rearranging by placing the second order terms to the left and the rest on the right side become:

$$\begin{aligned} aa\ddot{A}+ba\ddot{B}+ca\ddot{C}+da\ddot{D}+ea\ddot{E}+fa\ddot{F} = & -(a_1a_1\dot{A}+a_2a_2\dot{A}+b_1a_1\dot{B}+b_2a_2\dot{B} \\ & + c_1a_1\dot{C}+c_2a_2\dot{C}+d_1a_1\dot{D}+d_2a_2\dot{D} \\ & + e_1a_1\dot{E}+e_2a_2\dot{E}+f_1a_1\dot{F}+f_2a_2\dot{F}) \\ & + IF1 \end{aligned}$$

$$\begin{aligned} ab\ddot{A}+bb\ddot{B}+cb\ddot{C}+db\ddot{D}+eb\ddot{E}+fb\ddot{F} = & -(a_1b_1\dot{A}+a_2b_2\dot{A}+b_1b_1\dot{B}+b_2b_2\dot{B} \\ & + c_1b_1\dot{C}+c_2b_2\dot{C}+d_1b_1\dot{D}+d_2b_2\dot{D} \\ & + e_1b_1\dot{E}+e_2b_2\dot{E}+f_1b_1\dot{F}+f_2b_2\dot{F}) \\ & + IF2 \end{aligned}$$

$$\begin{aligned} ac\ddot{A}+bc\ddot{B}+cc\ddot{C}+dc\ddot{D}+ec\ddot{E}+fc\ddot{F} = & -(a_1c_1\dot{A}+a_2c_2\dot{A}+b_1c_1\dot{B}+b_2c_2\dot{B} \\ & + c_1c_1\dot{C}+c_2c_2\dot{C}+d_1c_1\dot{D}+d_2c_2\dot{D} \\ & + e_1c_1\dot{E}+e_2c_2\dot{E}+f_1c_1\dot{F}+f_2c_2\dot{F}) \\ & + IF3 \end{aligned}$$

$$\begin{aligned} ad\ddot{A}+bd\ddot{B}+cd\ddot{C}+dd\ddot{D}+ed\ddot{E}+fd\ddot{F} = & -(a_1d_1\dot{A}+a_2d_2\dot{A}+b_1d_1\dot{B}+b_2d_2\dot{B} \\ & + c_1d_1\dot{C}+c_2d_2\dot{C}+d_1d_1\dot{D}+d_2d_2\dot{D} \\ & + e_1d_1\dot{E}+e_2d_2\dot{E}+f_1d_1\dot{F}+f_2d_2\dot{F}) \\ & + IF4 \end{aligned}$$

$$\begin{aligned} ae\ddot{A}+be\ddot{B}+ce\ddot{C}+de\ddot{D}+ee\ddot{E}+fe\ddot{F} = & -(a_1e_1\dot{A}+a_2e_2\dot{A}+b_1e_1\dot{B}+b_2e_2\dot{B} \\ & + c_1e_1\dot{C}+c_2e_2\dot{C}+d_1e_1\dot{D}+d_2e_2\dot{D} \\ & + e_1e_1\dot{E}+e_2e_2\dot{E}+f_1e_1\dot{F}+f_2e_2\dot{F}) \\ & + IF5 \end{aligned}$$

$$af\ddot{A}+bf\ddot{B}+cf\ddot{C}+df\ddot{D}+ef\ddot{E}+ff\ddot{F} = -(a_1\dot{f}_1\dot{A}+a_2\dot{f}_2\dot{A}+b_1\dot{f}_1\dot{B}+b_2\dot{f}_2\dot{B} \\ + c_1\dot{f}_1\dot{C}+c_2\dot{f}_2\dot{C}+d_1\dot{f}_1\dot{D}+d_2\dot{f}_2\dot{D} \\ + e_1\dot{f}_1\dot{E}+e_2\dot{f}_2\dot{E}+f_1\dot{f}_1\dot{F}+f_2\dot{f}_2\dot{F}) \\ + IF6$$

where $\ddot{A}=\dot{u}$, $\dot{A}=u$, $\ddot{B}=\dot{v}$, $\dot{B}=v$, $\ddot{C}=\dot{w}$, $\dot{C}=w$, $\ddot{D}=\dot{p}$, $\dot{D}=p$

$\ddot{E}=\dot{q}$, $\dot{E}=q$, $\ddot{F}=\dot{r}$, $\dot{F}=r$, terms IF include all non-linear terms such as wave force, wind, rudder deflection---etc.

In the six equations, the right can be set equal to

I_1, I_2, \dots, I_6 respectively, thus:

$$I_1 = -(a_1\dot{a}_1\dot{A}+a_2\dot{a}_2\dot{A}+b_1\dot{a}_1\dot{B}+ \dots -f_2\dot{a}_2\dot{F}) + IF1$$

$$I_2 = -(a_1\dot{b}_1\dot{A}+a_2\dot{b}_2\dot{A}+b_1\dot{b}_1\dot{B}+ \dots -f_2\dot{b}_2\dot{F}) + IF2$$

$$I_3 = -(a_1\dot{c}_1\dot{A}+a_2\dot{c}_2\dot{A}+b_1\dot{c}_1\dot{B}+ \dots -f_2\dot{c}_2\dot{F}) + IF3$$

$$I_4 = -(a_1\dot{d}_1\dot{A}+a_2\dot{d}_2\dot{A}+b_1\dot{d}_1\dot{B}+ \dots -f_2\dot{d}_2\dot{F}) + IF4$$

$$I_5 = -(a_1\dot{e}_1\dot{A}+a_2\dot{e}_2\dot{A}+b_1\dot{e}_1\dot{B}+ \dots -f_2\dot{e}_2\dot{F}) + IF5$$

$$I_6 = -(a_1\dot{f}_1\dot{A}+a_2\dot{f}_2\dot{A}+b_1\dot{f}_1\dot{B}+ \dots -f_2\dot{f}_2\dot{F}) + IF6$$

the equations then have the form that follows,

$$aa\ddot{A} + ba\ddot{B} + ca\ddot{C} + da\ddot{D} + ea\ddot{E} + fa\ddot{F} = I_1$$

$$ab\ddot{A} + bb\ddot{B} + cb\ddot{C} + db\ddot{D} + eb\ddot{E} + fb\ddot{F} = I_2$$

$$ac\ddot{A} + bc\ddot{B} + cc\ddot{C} + dc\ddot{D} + ec\ddot{E} + fc\ddot{F} = I_3$$

$$ad\ddot{A} + bd\ddot{B} + cd\ddot{C} + dd\ddot{D} + ed\ddot{E} + fd\ddot{F} = I_4$$

$$ae\ddot{A} + be\ddot{B} + ce\ddot{C} + de\ddot{D} + ee\ddot{E} + fe\ddot{F} = I_5$$

$$af\ddot{A} + bf\ddot{B} + cf\ddot{C} + df\ddot{D} + ef\ddot{E} + ff\ddot{F} = I_6$$

expressing in matrix form:

$$\begin{vmatrix} aa & ba & ca & da & ea & fa \\ ab & bb & cb & db & eb & fb \\ ac & bc & cc & dc & ec & fc \\ ad & bd & cd & dd & ed & fd \\ ae & be & ce & de & ee & fe \\ af & bf & cf & df & ef & ff \end{vmatrix} \begin{vmatrix} \ddot{A} \\ \ddot{B} \\ \ddot{C} \\ \ddot{D} \\ \ddot{E} \\ \ddot{F} \end{vmatrix} = \begin{vmatrix} I_1 \\ I_2 \\ I_3 \\ I_4 \\ I_5 \\ I_6 \end{vmatrix}$$

Apply Cramer's rule to solve for \ddot{A} , \ddot{B} ----- \ddot{F} in terms of I_1 -- I_6

$$\ddot{A} = \frac{\begin{vmatrix} I_1 & ba & ca & da & ea & fa \\ I_2 & bb & cb & db & eb & fb \\ I_3 & bc & cc & dc & ec & fc \\ I_4 & bd & cd & dd & ed & fd \\ I_5 & be & ce & de & ee & fe \\ I_6 & bf & cf & df & ef & ff \end{vmatrix}}{\begin{vmatrix} aa & ba & ca & da & ea & fa \\ ab & bb & cb & db & eb & fb \\ ac & bc & cc & dc & ec & fc \\ ad & bd & cd & dd & ed & fd \\ ae & be & ce & de & ee & fe \\ af & bf & cf & df & ef & ff \end{vmatrix}}$$

define the denominator determinant $\overset{\Delta}{=} \Delta$ and for the nominator let cofactor of $I_1 \overset{\Delta}{=} \text{cof. } aa$
cofactor of $I_2 \overset{\Delta}{=} \text{cof. } ab$, -----and cofactor of $I_6 \overset{\Delta}{=} \text{cof. } af$ equations becomes:

$$\ddot{A} = \frac{(\text{cof. } aa I_1 + \text{cof. } ab I_2 + \text{cof. } ac I_3 + \text{cof. } ad I_4 + \text{cof. } ae I_5 + \text{cof. } af I_6)}{\Delta}$$

In the same way solve for B, C, -----

$$\ddot{B} = \frac{(\text{cof } baI_1 \text{ cof } bbI_2 \text{ cof } bcI_3 \text{ cof } bdI_4 \text{ cof } beI_5 \text{ cof } bfI_6)}{\Delta}$$

$$\ddot{C} = \frac{(\text{cof } caI_1 \text{ cof } cbI_2 \text{ cof } ccI_3 \text{ cof } cdI_4 \text{ cof } ceI_5 \text{ cof } cfI_6)}{\Delta}$$

$$\ddot{D} = \frac{(\text{cof } daI_1 \text{ cof } dbI_2 \text{ cof } dcI_3 \text{ cof } ddI_4 \text{ cof } deI_5 \text{ cof } dfI_6)}{\Delta}$$

$$\ddot{E} = \frac{(\text{cof } eaI_1 \text{ cof } ebI_2 \text{ cof } ecI_3 \text{ cof } edI_4 \text{ cof } eeI_5 \text{ cof } efI_6)}{\Delta}$$

$$\ddot{F} = \frac{(\text{cof } faI_1 \text{ cof } fbI_2 \text{ cof } fcI_3 \text{ cof } fdI_4 \text{ cof } feI_5 \text{ cof } ffI_6)}{\Delta}$$

Then the value of \dot{A} , A , \dot{B} , B ----- \dot{F} , F by integration such that

$$\dot{A} = \frac{dA}{dt} = \int \frac{d^2A}{dt^2} , \quad A = \int \frac{dA}{dt}$$

A block diagram to compute all of the variables in the set of equations is presented in Fig. 2.

In the computer program that is used for simulation all six equations for six degrees of freedom are used, but are interested in less than six degrees of freedom. The same program can be used by setting the coupling terms of non-used equations equal to zero and one in terms of principal diagonal, e.g. only three degrees of freedom are used in this study, surge, sway and yaw, then all coupling terms

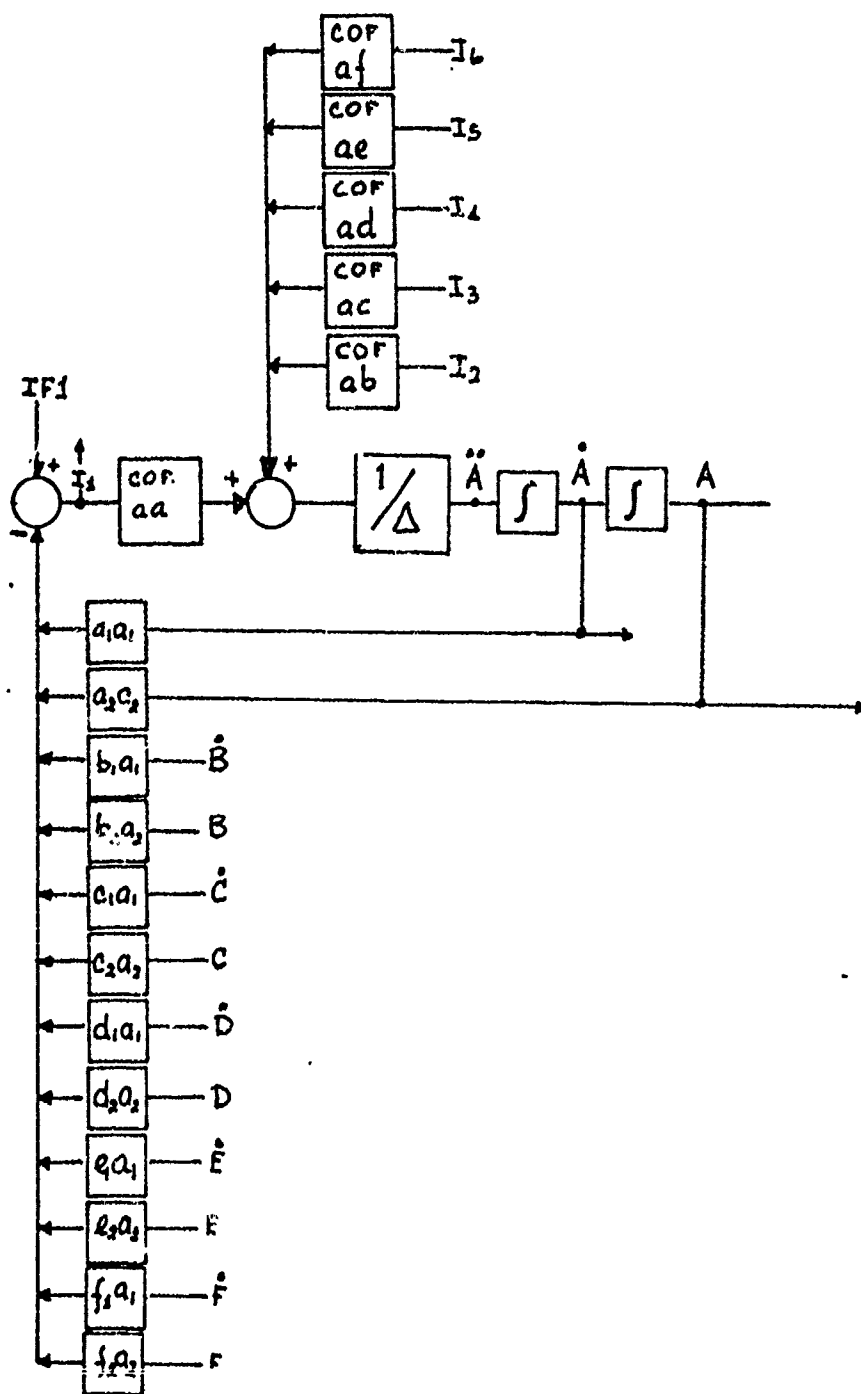


Fig. 2
TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREES OF FREEDOM(Eqn.A)

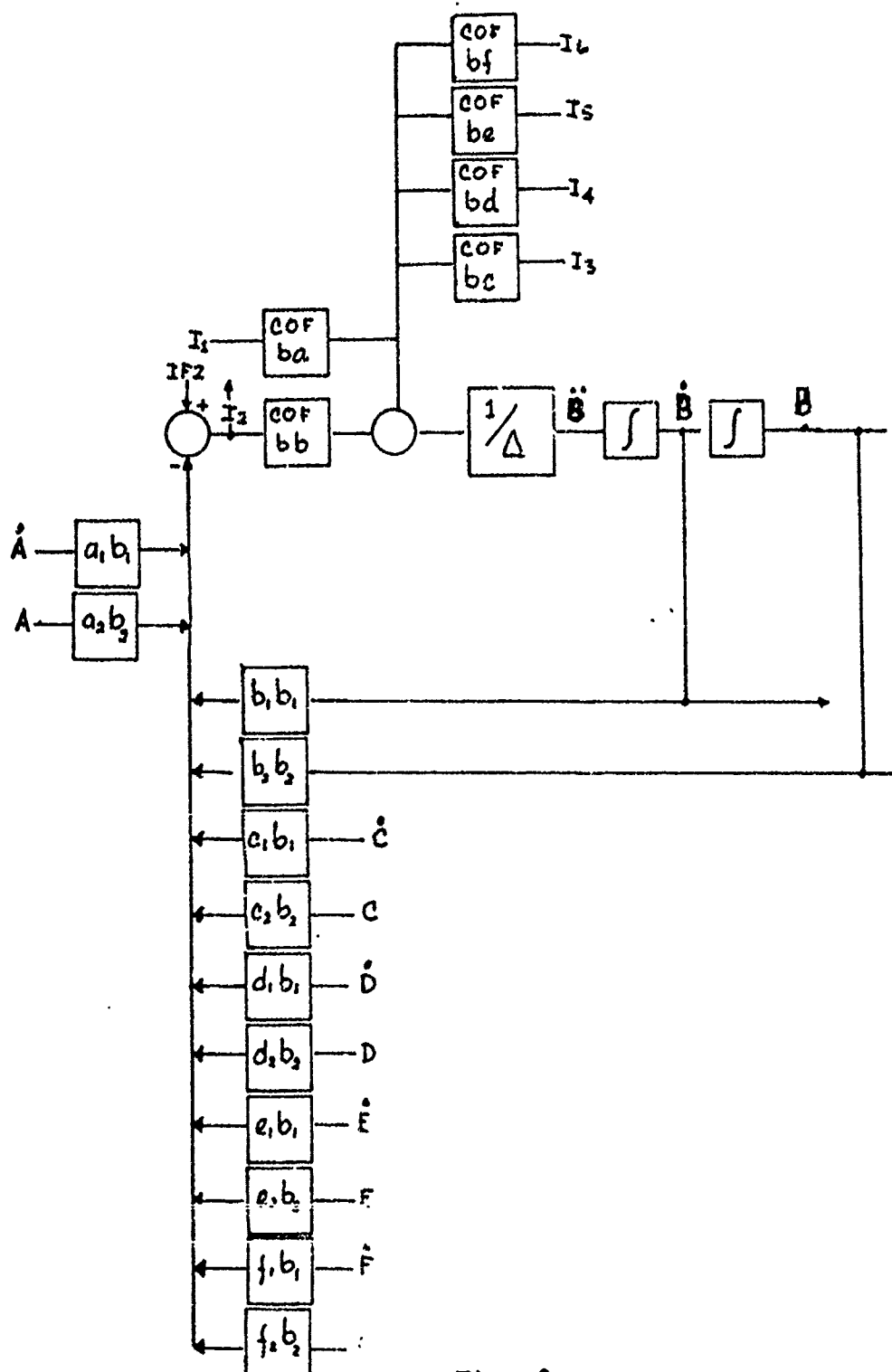


Fig. 2
TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREES OF FREEDOM (Eqn. B)

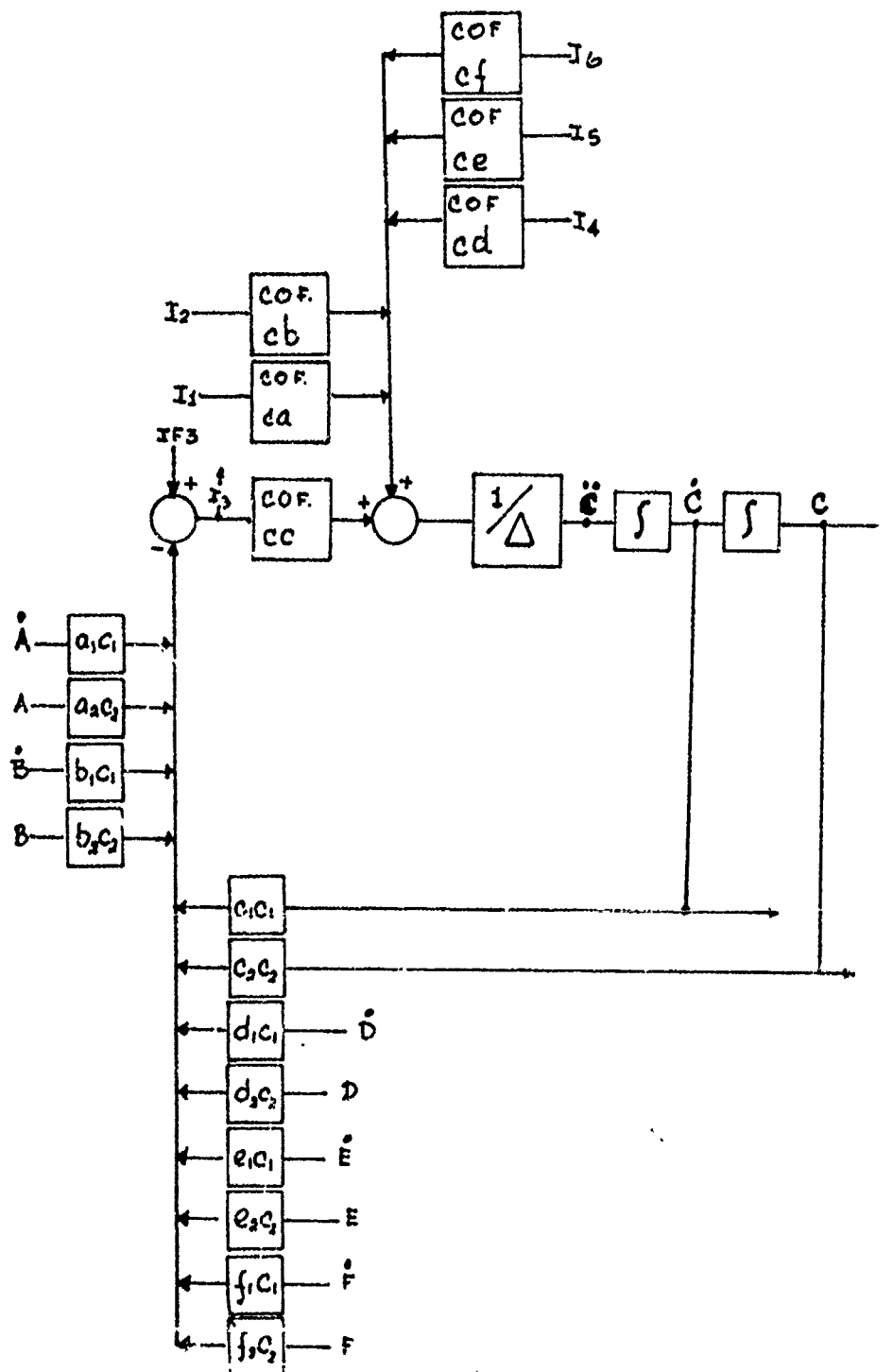


Fig. 2

TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREES OF FREEDOM(Eqn.C)

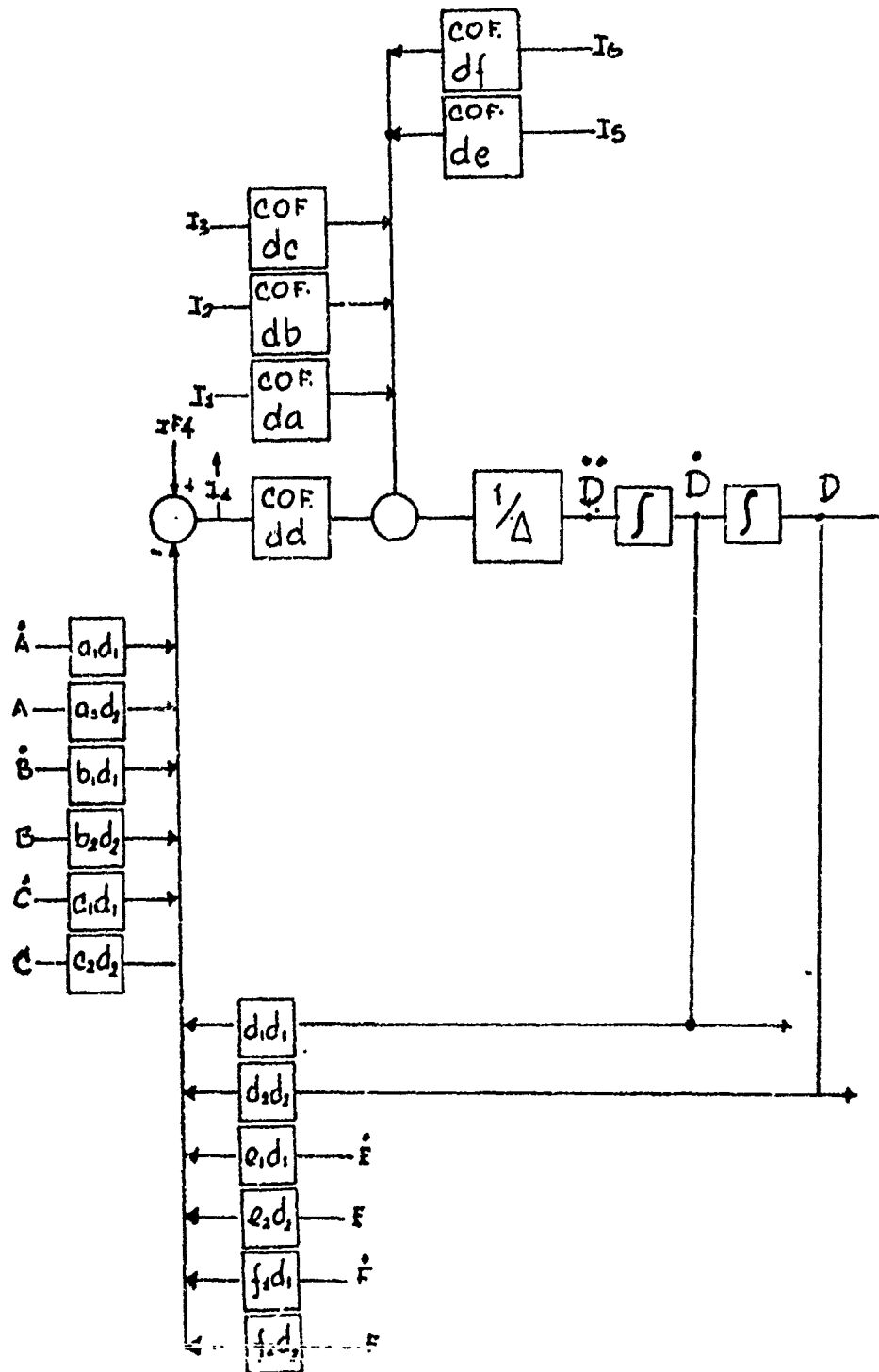
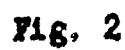


Fig. 2

TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREE OF FREEDOM(Eqn.D)



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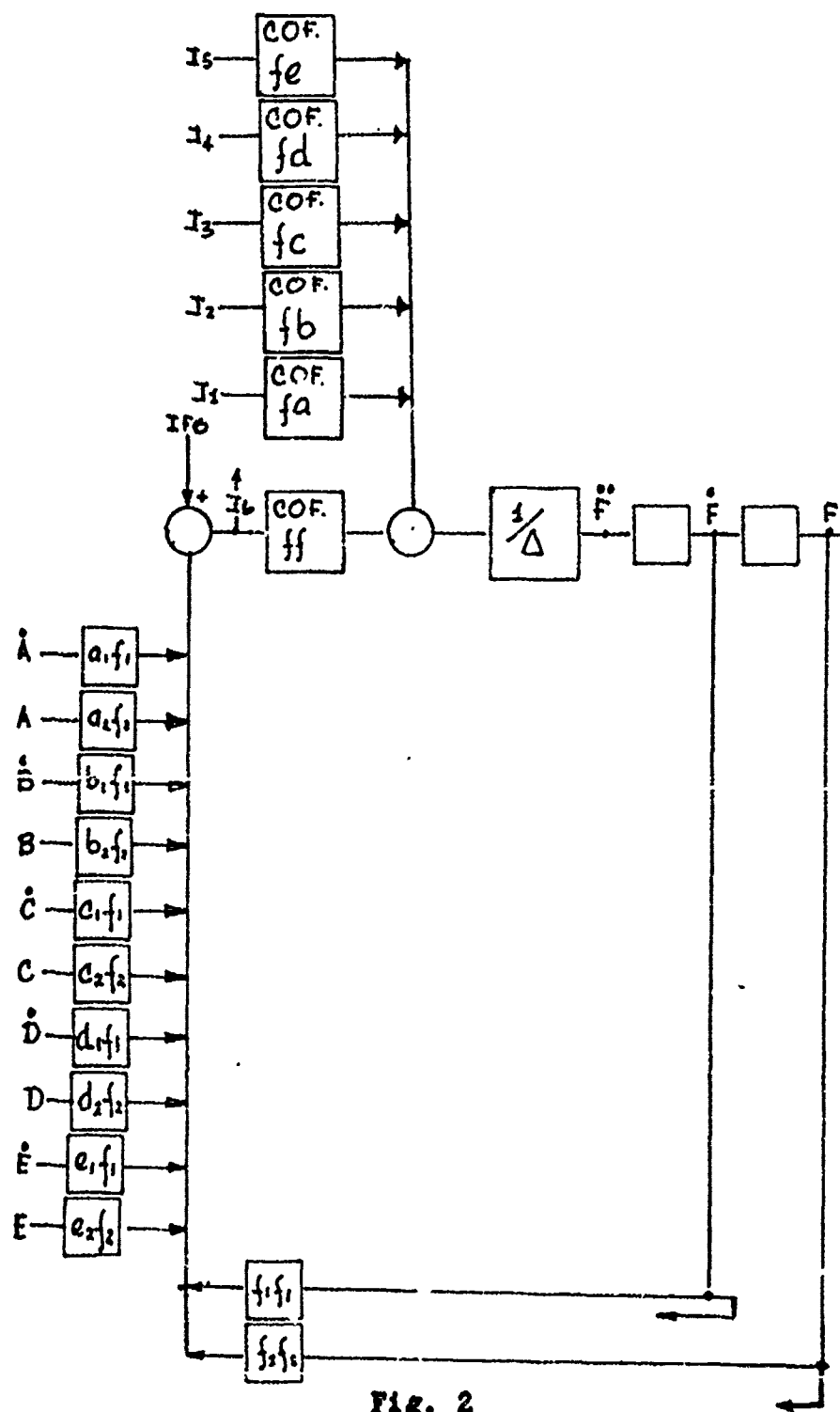


Fig. 2

TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREES OF FREEDOM(Eqn.F)

are set equal to zero and unused terms on the principal diagonal equal to one, for example :

$$\begin{vmatrix}
 aa & ba & 0 & 0 & 0 & fa \\
 ab & bb & 0 & 0 & 0 & fb \\
 0 & 0 & 1 & 0 & 0 & 0 \\
 0 & 0 & 0 & 1 & 0 & 0 \\
 0 & 0 & 0 & 0 & 1 & 0 \\
 at & bf & 0 & 0 & 0 & ff
 \end{vmatrix}
 =
 \begin{vmatrix}
 \ddot{A} \\
 \ddot{B} \\
 \ddot{C} \\
 \ddot{D} \\
 \ddot{E} \\
 \ddot{F}
 \end{vmatrix}
 =
 \begin{vmatrix}
 I_1 \\
 I_2 \\
 I_3 \\
 I_4 \\
 I_5 \\
 I_6
 \end{vmatrix}$$

and the left side of the unused equations are set equal to zero.

With this program non-linear terms can be added such as, rudder deflection of waves and wind, etc., which will be done by adding all of these whose sum is IF_N e.g.

$$1F_1 = KA_1 \times Dr + KA_2 \times Ds + KA_3 \times Db + NA$$

where Dr, Ds and Db are rudder deflection, canard deflection..... etc. NA is the sum of all non-linear terms that effect the surge equation (X equation).

The program that will be used for solving these equations is the "Continuous Systems Modeling Program" (CSMP) [Ref. 3] in which all constants are declared in the first section and then set the value of matrix for aa, ab, ac and so on (in program AAA is used for aa, AAB for ab AFF for ff). In the initial section values of the COFACTORS aa, ba....are determined. All of the COFACTORS and the subprogram VALUE is used to compute. This subprogram finds the value of the determinant of the

matrix. For all of the COFACTOR terms the element is set equal to one and the rest of the elements in that row and column are set equal to zero. For example, to find the value of COF.aa the following array is obtained:

$$\text{COF aa} = \begin{vmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & \text{bb} & \text{cb} & \text{db} & \text{eb} & \text{fb} \\ 0 & \text{bc} & \text{cc} & \text{dc} & \text{ec} & \text{fc} \\ 0 & \text{bd} & \text{cd} & \text{dd} & \text{ed} & \text{fd} \\ 0 & \text{be} & \text{ce} & \text{de} & \text{ee} & \text{fe} \\ 0 & \text{bf} & \text{cf} & \text{df} & \text{ef} & \text{ff} \end{vmatrix}$$

(In the computer program BAP is used for $a_1 a_1$, GAA for $a_2 a_2 \dots$). After the value of Δ and all cofactors are determined, the dynamic section is used to determine BAA, BABGAA, GAB (if those terms contain variables).

In the dynamic section all variables that are functions of time are determined. The defining relations of the variables are also included in the dynamics section, i.e. $\text{UDOT} = \text{ADDOT} (\dot{U} = \ddot{A})$, $U = \text{ADOT} \dots \dots \dots \text{etc.}$ XH,YH,ZH are determined and are the vector terms, X,Y,Z whose origin is fixed on the earth (relative to the earth).

III. STUDY OF SHIP "D" PERFORMANCE

In this section the computer will be used to solve the equation of motion describing ship "D". The hydrodynamic coefficients and constants that were obtained from NSRDC (NAVAL SHIP RESEARCH AND DEVELOPMENT CENTER) [Ref. 6] for this study will concern only three degrees of freedom such that SURGE, SWAY, and YAW.

Equations of motion of the ship (nondimensional)

Axial Force

$$\begin{aligned} m(\dot{u}-vr+wg) = & \frac{\rho}{2} \ell (X_{gg}g^2 + X_{rr}r^2 + X_{rp}rp) \\ & + \frac{\rho}{2} (X_{\dot{u}}\dot{u} + X_{vr}vr + X_{wg}wg) \\ & + \frac{\rho}{2\ell} (X_{vv}v^2) \\ & + \frac{\rho}{2\ell} u^2 (X_{\delta r\delta r}\delta r^2 + X_{\delta s\delta s}\delta s^2 + X_{\delta b\delta b}\delta b^2) \\ & + \frac{\rho}{2\ell} (A_1 u^2 + A_2 u \cdot u_c + A_3 u_c^2) \end{aligned}$$

Lateral Force

$$\begin{aligned} m(\dot{v}+ur-wp) = & \frac{\rho}{2} \ell (Y_{\dot{p}}\dot{p} + Y_{\dot{r}}\dot{r} + Y_{pq}pq) \\ & + \frac{\rho}{2} (Y_{wp}wp + Y_{v|r}|v|r| + uY_{r}r + Y_{\dot{v}}\dot{v} + uY_{p}p) \\ & + \frac{\rho}{2} (uY_{v}v + Y_{wv}wv + Y_{|v|v}|v|v|) \\ & + \frac{\rho}{2\ell} u^2 (Y_{\delta r}\delta r) \end{aligned}$$

Yawing Moment

$$\begin{aligned} I_z \dot{r} + (I_Y - I_X)pq = & \frac{\rho}{2} (N_{\dot{r}}\dot{r} + N_{\dot{p}}\dot{p} + N_{pq}pq) \\ & + \frac{\rho}{2\ell} (N_{\dot{v}}\dot{v} + uN_{p}p + uN_{r}r + N_{wp}wp + N_{|v|r}|v|r|) \end{aligned}$$

$$+ \frac{\rho}{2\ell^2} (u N_v v + N_{wv} w v + N_{|v|} |v| |v|) \\ + \frac{\rho}{2\ell^2} u^2 (N_{\delta r} \delta r)$$

ρ , the density of fluid is taken as 2 and the terms including w , p , q (heave, roll and pitch) are set equal to zero.

The nonlinear terms such as the squared terms and product terms of v and r are omitted initially. This is in agreement with the small perturbation theory.

After rearranging, the equations become

$$(X_{\dot{u}} - m) \dot{u} = -X_{\delta r} \delta r^2 \frac{u^2}{\ell} \\ (Y_{\dot{v}} - m) \dot{v} + Y_{\dot{r}} \dot{r} = -\frac{u}{\ell} Y_v v - u Y_r r - \frac{u^2}{\ell} Y_{\delta r} \delta r \\ N_v \frac{\dot{v}}{\ell} + (N_r - I_z) \dot{r} = -\frac{u}{\ell^2} N_v v - \frac{u}{\ell^2} N_r r - \frac{u^2}{\ell^2} N_{\delta r} \delta r$$

Set the left side of the equations equal to I_1, \dots, I_6 , then the matrix equation becomes

$$\begin{vmatrix} (X_{\dot{u}} - m) & 0 & 0 & 0 & 0 & 0 \\ 0 & (Y_{\dot{v}} - m) & 0 & 0 & 0 & Y_{\dot{r}} \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & N_v/\ell & 0 & 0 & 0 & (N_r - I_z) \end{vmatrix} \begin{vmatrix} \ddot{u} \\ \ddot{v} \\ \ddot{r} \\ \ddot{r} \\ \ddot{r} \\ \ddot{r} \end{vmatrix} = \begin{vmatrix} I_1 \\ I_2 \\ I_3 \\ I_4 \\ I_5 \\ I_6 \end{vmatrix}$$

where $\dot{u} = \ddot{A}$, $\dot{v} = \ddot{B}$, $\dot{r} = \ddot{F}$.

Also set the right side of the equations equal to I_1

----- I_6

$$\begin{array}{c} \left| \begin{array}{c} I_1 \\ I_2 \\ I_3 \\ I_4 \\ I_5 \\ I_6 \end{array} \right| = \left| \begin{array}{cccccc} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & \frac{u}{\ell} Y_v & 0 & 0 & 0 & u Y_r \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & \frac{u}{\ell^2} N_v & 0 & 0 & 0 & \frac{u}{\ell} N_r \end{array} \right| \left| \begin{array}{c} \dot{A} \\ \dot{B} \\ \dot{C} \\ \dot{D} \\ \dot{E} \\ \dot{F} \end{array} \right| \end{array}$$

+

All Zero

$$\begin{array}{c} \left| \begin{array}{c} A \\ B \\ C \\ D \\ E \\ F \end{array} \right| + \left| \begin{array}{c} IF1 \\ IF2 \\ IF3 \\ IF4 \\ IF5 \\ IF6 \end{array} \right| \end{array}$$

where $u = \dot{A}$, $v = \dot{B}$, $r = \dot{F}$, and $IF1 = -X_{\delta r} \delta r \frac{u^2}{\ell^2}$

$$IF2 = -Y_{\delta r} \delta r \frac{u^2}{\ell^2}$$

$$IF3 = 0$$

$$IF4 = 0$$

$$IF5 = 0$$

$$IF6 = -N_{\delta r} \delta r \frac{u^2}{\ell^2}$$

A. NONLINEAR TERMS NOT INCLUDED

TABLE I

Hydrodynamic Coefficients and Constants of Ship "D"
for Linear Terms (Non-dimensional)

m	$=$	0.0045
I_z	$=$	0.0003
N_r	$=$	0.0012
N_r	$=$	-0.0002
N_v	$=$	-0.0012
N_v	$=$	-0.0001
X_u	$=$	-0.00036
Y_v	$=$	-0.0025
Y_v	$=$	-0.0063
Y_r	$=$	0.004
$X_{\delta r \delta r}$	$=$	-0.0011
$Y_{\delta r}$	$=$	0.0019
$N_{\delta r}$	$=$	-0.00084

δs and δb equal zero

In the computer program, set all coefficients in section 1 and set $AAA = X_u - m$, $AAB = 0$ ---- $AFF = N_r - I_z$. After this, use subprogram value to find the determinant and coefficient of AA, BB ---- and then set $BBB = \frac{u}{l} Y_v$, $BFB = u Y_r$ ----- $BFF = \frac{u}{l} N_r$.

$$\begin{aligned} IF1 &= KA1 \delta r \quad \text{where } KA1 = -X_{\delta r \delta r} \delta r \frac{u^2}{l} \\ IF2 &= KB1 \delta r \quad \text{where } KB2 = -Y_{\delta r} u^2 / l \\ IF3 &= KF1 \delta r \quad \text{where } KF1 = -N_{\delta r} u^2 / l^2 \end{aligned}$$

Following is the program that used CSMP to determine the turning circles for rudder angles of 15° (-0.2619 rad.), 25° (-0.4365 rad.) and 35° (-0.6111 rad.). Result of this study is presented in Fig. 3. The turning rate as a function of time is interested and the results of this analysis are presented in Fig. 4. The ships turning performance expressed in transfer ship lengths as time is shown in Fig. 5 and the heading angle as a function of time is given in Fig. 6. Fig. 7 shows the results of the zig-zag maneuver, curve shown the yaw angle and rudder angle in degree as a function of time, for this study the same program was used, but set DR in dynamic section:

```
DR= -0.06984(RAMP(0.0)-RAMP(5.0))+0.04056(RAMP(40.0)...
    -RAMP(65.0)-RAMP(145.0)+RAMP(170.0)+RAMP(250.0)....
    -RAMP(265.0)-RAMP(345.0)+RAMP(425.0)+RAMP(440.0))
```

and use prepare statement prepare X, YAWD and prepare X,
DOO (YAWD = YAW* 57.273, DOO = DR* 57.273).


```

PARAM YVDCT=-C.CC025
PARAM YV=-0.0063
PARAM LC=1.0
PARAM YRDCI=-C.CC02
PARAM YR=0.004
PARAM NVDCI=-C.CC01
PARAM NVDCI=-C.CC02
PARAM IZ=0.0003
PARAM NV=-0.0012
PARAM NK=-0.0012
PARAM XL=-0.0012
PARAM KA2=0.
PARAM KA2=0.
PARAM KP2=0.
PARAM KP3=0.
PARAM KCI=0.
PARAM KC2=0.
PARAM KC3=0.
PARAM KL2=0.
PARAM KL3=0.
PARAM KL1=0.0
PARAM KE2=0.
PARAM KE3=0.
*SECTION 2 - PARAMETERS CALCULATIONS
INITIAL
*SECTION 2A - ALL PARAMETERS MUST BE DEFINED AND IN SEQUENCE:
**AAC, AAC, AAE, AAF, ABA, ABB, ABC, ABC, ABE, AEF, ACA, ACB, ACC, ACC, ACE
**ACF, ACA, ADE, ADE, ADE, AEA, AEB, AEC, AED, AEE, AEF, AFA, AFB, AFC, AFC, AFE, AFE, AFF
**ACF, AAA=XLDCI-ML
AAE=0.
AAC=0.
AAC=0.
AAE=0.
AAF=0.
AEB=0.
AEB=YVDCI-ML
ABC=0.
ABC=0.0
ABE=0.
ABE=NVDCI/LC
ACA=0.
ACB=0.
ACC=1.0
ACC=0.
ACE=0.
ACE=0.
ACA=0.0
ACB=0.0

```

```

CC=0.0
ACC=1.0
ACE=0.0
ACF=0.0
ACH=0.0
AEC=0.0
AED=0.0
AEE=1.0
AEF=0.0
AFB=LC*YRDGT
AFC=0.0
AFD=0.0
AFF=0.0
AFF=NRDCT-12
SECTION 2A
LEL=VALUE(AAA,0,0)
CCFAL=VALUE(AAA,1,1)
CCFAC=VALUE(AAA,2,1)
CCFAD=VALUE(AAA,3,1)
CCFAE=VALUE(AAA,4,1)
CCFAF=VALUE(AAA,5,1)
CCFBH=VALUE(AAA,6,1)
CCFBG=VALUE(AAA,2,2)
CCFBC=VALUE(AAA,3,2)
CCFBD=VALUE(AAA,4,2)
CCFBE=VALUE(AAA,5,2)
CCFBI=VALUE(AAA,6,2)
CCFCA=VALUE(AAA,1,3)
CCFCB=VALUE(AAA,2,3)
CCFCG=VALUE(AAA,3,3)
CCFCJ=VALUE(AAA,4,3)
CCFCI=VALUE(AAA,5,3)
CCFCA=VALUE(AAA,6,4)
CCFLU=VALUE(AAA,2,4)
CCFLJ=VALUE(AAA,3,4)
CCFLD=VALUE(AAA,4,4)
CCFDE=VALUE(AAA,5,4)
CCFCH=VALUE(AAA,6,4)
CCFEB=VALUE(AAA,1,5)
CCFEC=VALUE(AAA,3,5)
CCFED=VALUE(AAA,4,5)
CCFEF=VALUE(AAA,5,5)
CCFEF=VALUE(AAA,6,5)

```

#END


```

F=INTGRL(C,FDOOT)
*SECTION 5-OUTPUT CHARACTERISTICS
TIMER CELT=C.01,FINTIM=240.0,CUTDEL=1.0,PRDEL=1.0
PREPARE YH,XH
END
PARAM CR=-C.4365
END
PARAM CR=-O.6111
END
STCP

```

```

FUNCTION VALUE(Y,I,M)
COMMON DIMENSION X(6,6),Y(6,6)
DO 1 M1=1,6
DO 1 M2=1,6
X(M1,M2)=Y(M1,M2)
1 IF(I.EQ.0) GO TO ICC
X(1,1)=0.
X(1,2)=0.
X(1,3)=0.
X(1,4)=0.
X(1,5)=0.
X(1,6)=0.
X(2,M)=0.
X(3,M)=0.
X(4,M)=0.
X(5,M)=0.
X(6,M)=0.
X(1,M)=1.
CONTINUE
ICC WRITE(0,49) I,M DETERMINANT FOR CCF, I1,I1,':')
49 FORMAT(7,1,6)
DO 50 M1=1,6
WRITE(6,51)(X(M1,M2),M2=1,6)
51 FORMAT(1X,E13.6)
50 CONTINUE
N=6
LL=1,LL
LL 34 L=1,N
KF=0
Z=C,C
DO 12 X=L,N
IF(2-ABS(X(K,L)))11,12,12
11 Z=ABS(X(K,L))
KP=K

```

```

12 CONTINUE
13 IF(L-KP) 13,20,20
14 J=L,N
2 Z=X(L,J)
14 X(L,J)=X(KP,J)
20 CC=-DD
31 IF(L-N) 31,40,40
31 LPL=L+1
34 K=LPL,N
32 IF(X(K,L)) 32,34,32
32 RATIO=X(K,L)/X(L,L)
33 LPL=J J=LPL,N
33 X(K,J)=X(K,J)-RATIO*X(L,J)
34 CONTINUE
40 K=1,N
41 CC=DD*X(K,K)
DD=CC
VALUE=D
WRITE(6,52) I,M,VALUE
52 FORMAT(' ',COF,I1,I1,'=',E15.6)
53 RETURN
ENDJCB
END

```

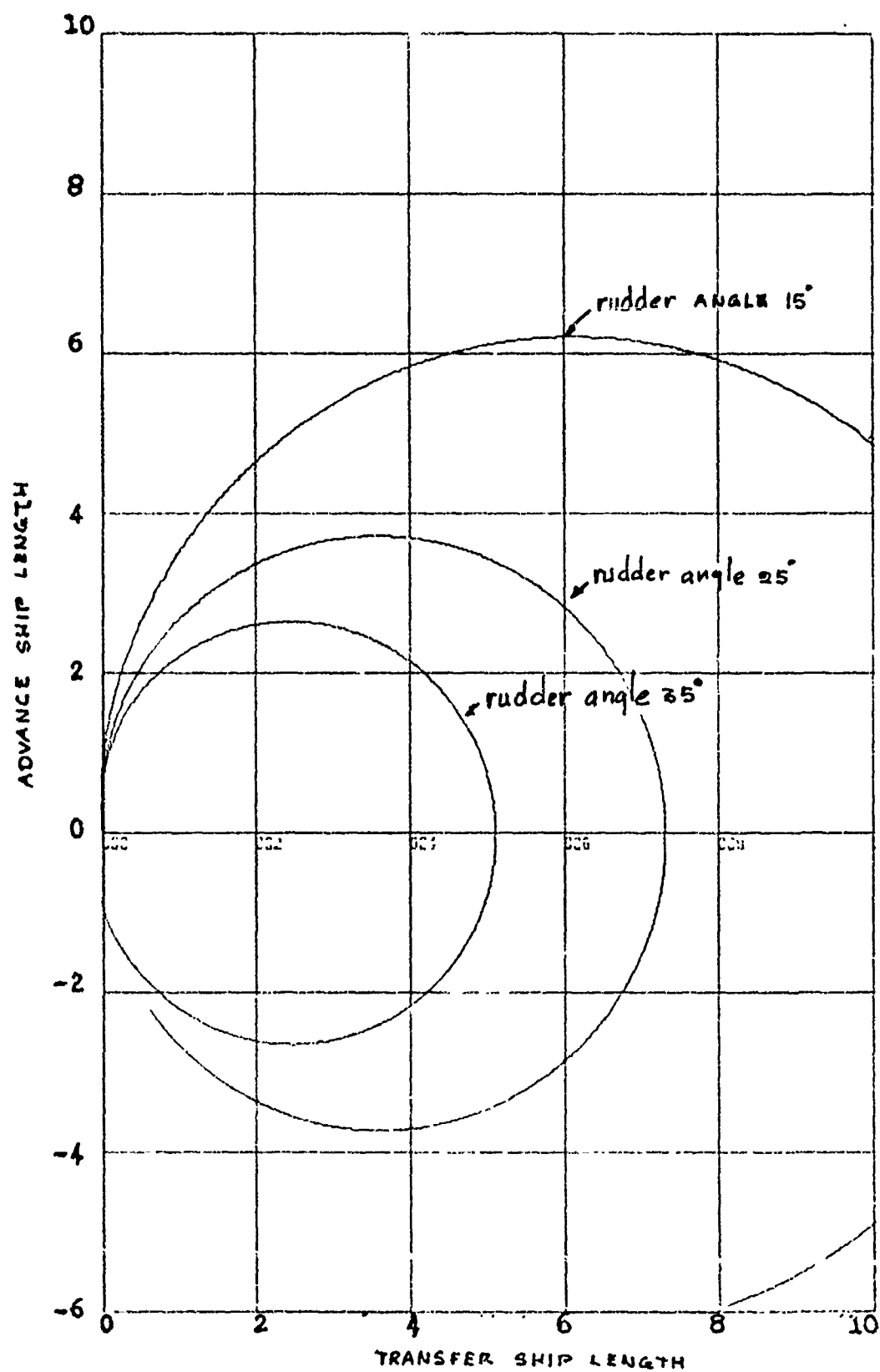


FIG. 3 ADVANCE VS. TRANSFER SHIP LENGTH
(RUDDER 15°, 25° & 35°)

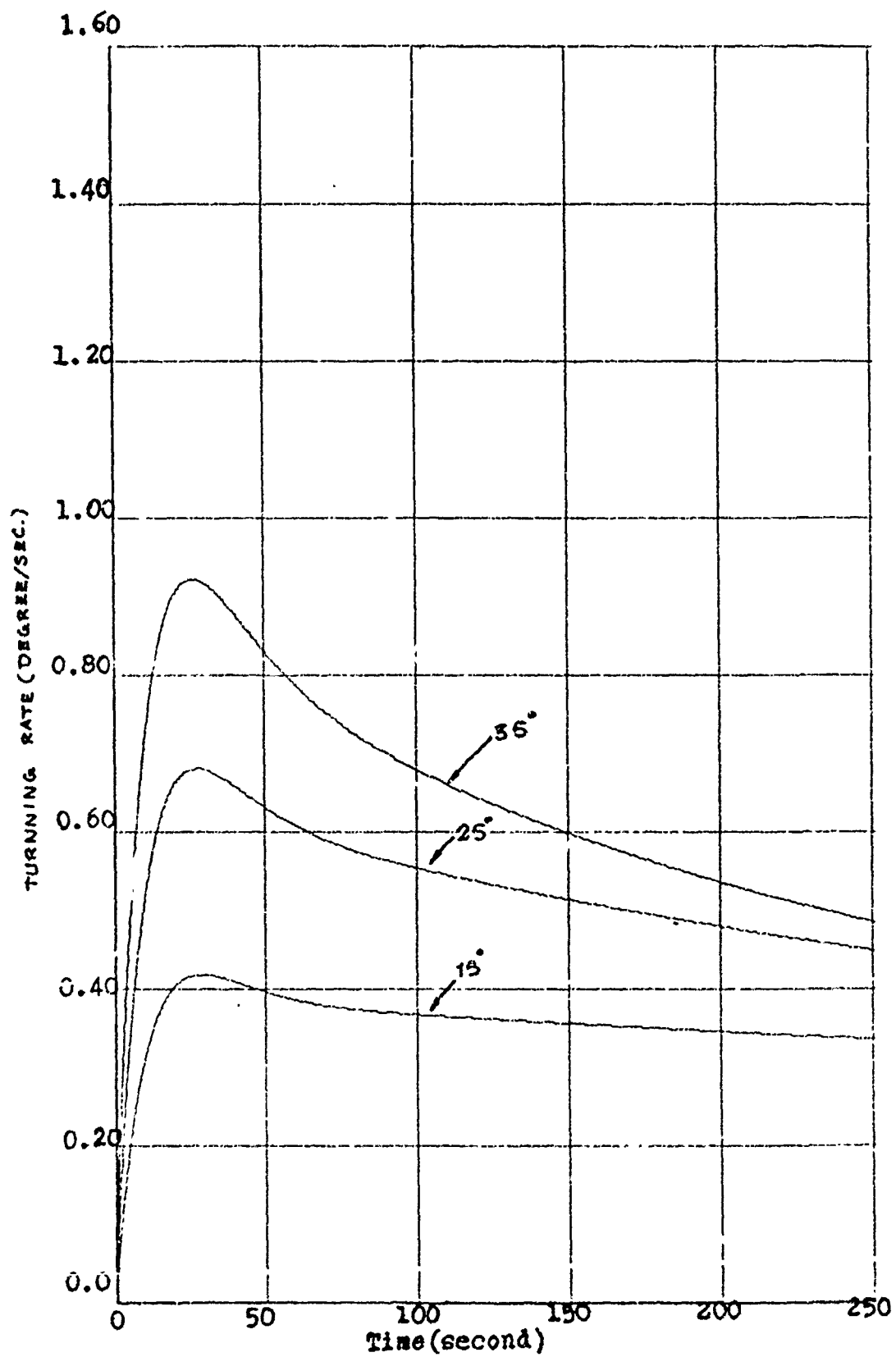


FIG. 4 TURNING RATE VS. TIME
(RUDDER 15°, 25°, 35°)

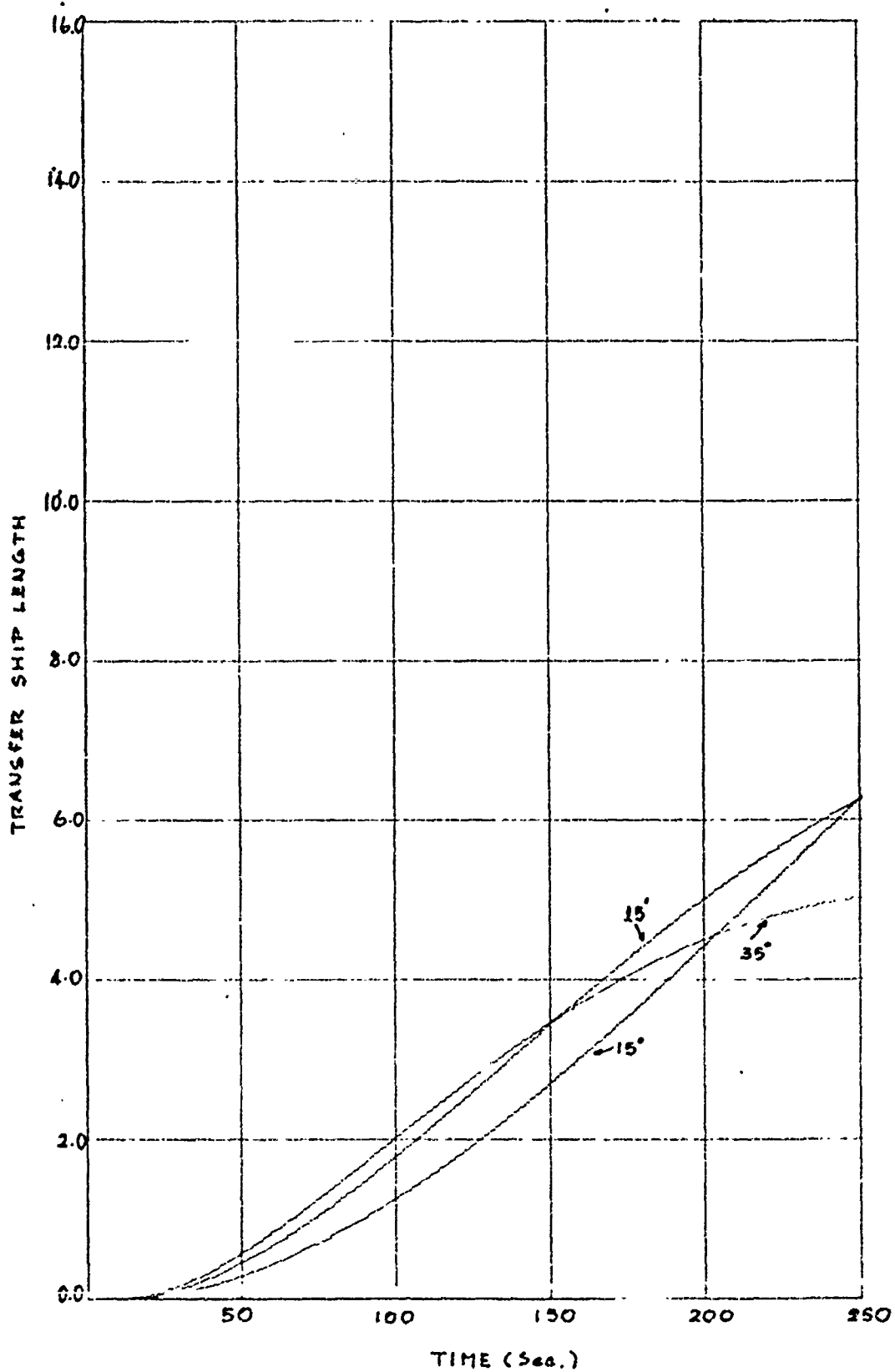


FIG. 5 TRANSFER SHIP LENGTH VS. TIME
(RUDDER ANGLE 15°, 25°, 35°)

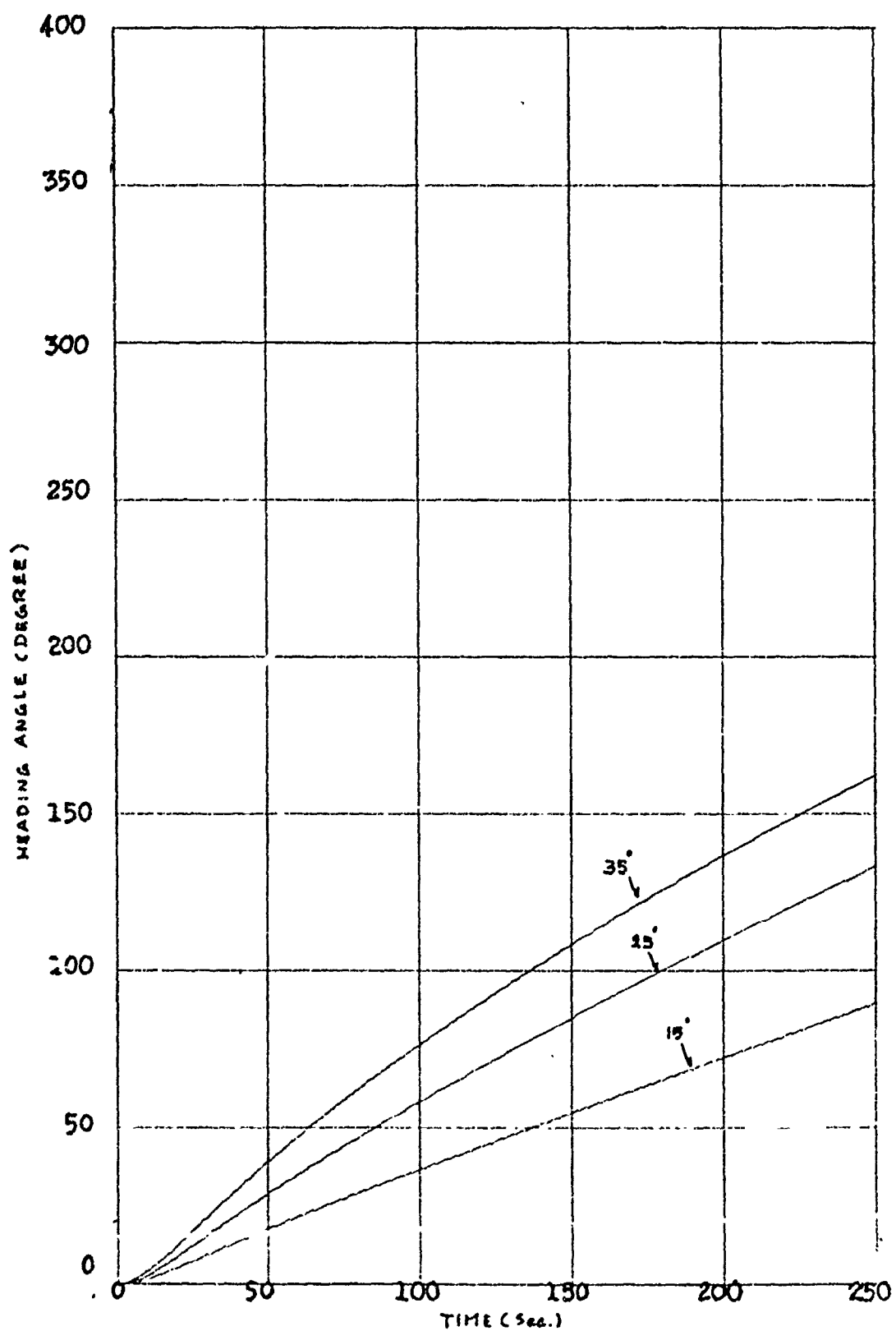


FIG. 6 HEADING ANGLE AS FUNCTION OF TIME
(RUDDER ANGLE 15°, 25° & 35°)

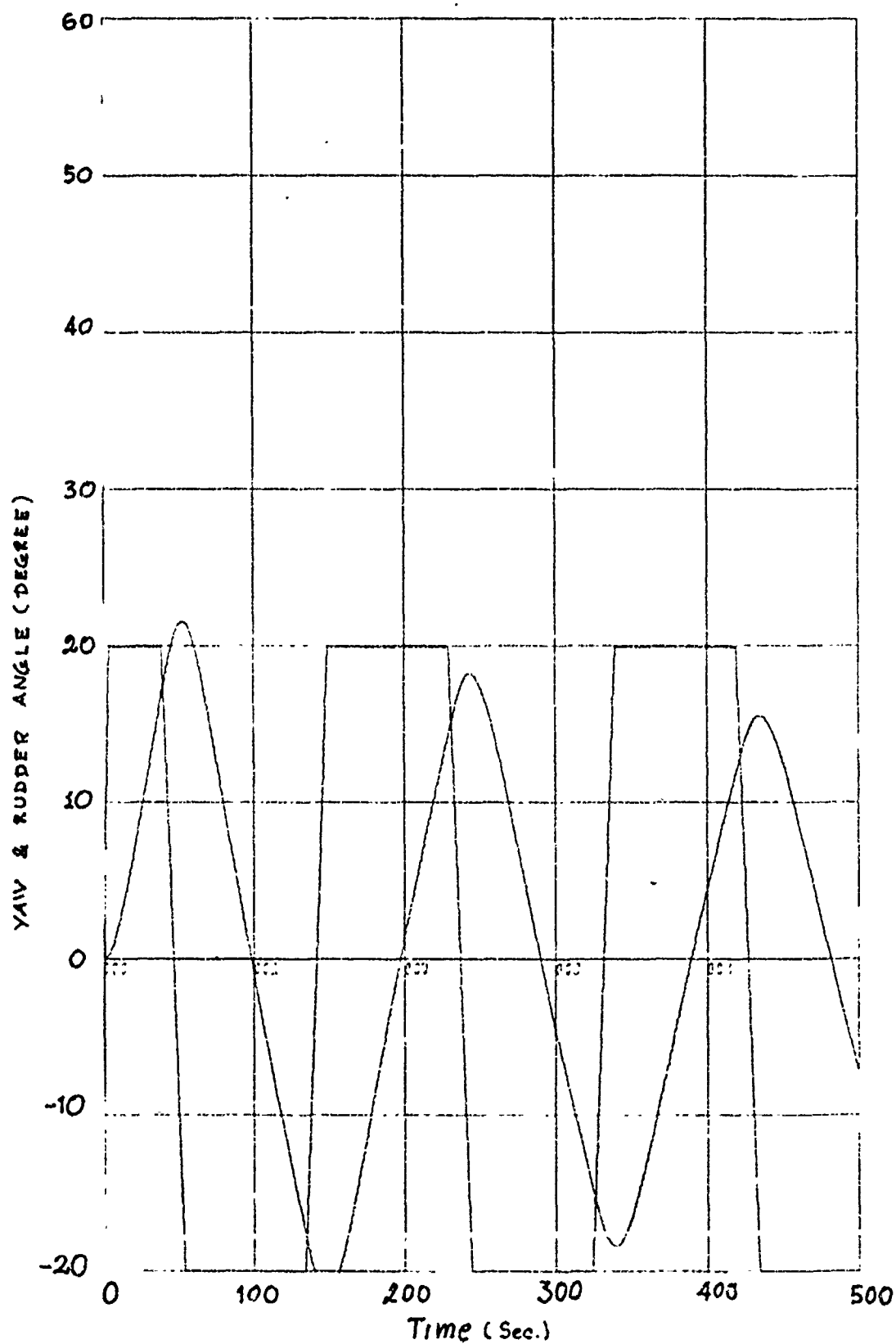


Fig. 7 YAW & RUDDER ANGLE VS. TIME ZIG-ZAG MANOEUVRE

E. INCLUDED NONLINEAR TERMS

The computer program 2 is the same as the computer program 1, but nonlinear terms are added by setting:

$$NA = NA1 + NA2 + NA3 + NA4$$

where

$$NA1 = -1(X_{qq}q^2 + X_{rr}r^2 + X_{rp}rp)$$

$$NA2 = -(mvr + X_{vr}vr + X_{wq}wq + mwq)$$

$$NA3 = -(X_{vv}v^2)/1$$

$$NA4 = -(A1u^2 + A2uu_c + A3u_c^2)/1$$

$$NB = NB1 + NB2 + NB3$$

where

$$NB1 = -1Y_{pq}pq$$

$$NB2 = -(Y_{wp}wp + Y_{v|r}|v|r| + mur + mwp)$$

$$NB3 = -(Y_{wv}wv + Y_{|v|v}|v|v|)/1$$

$$NF = NF1 + NF2 + NF3$$

where

$$NF1 = -N_{pq}pq + (Iy - Ix) pq$$

$$NF2 = -(N_{wp}wp + N_{|v|r}|v|r|)/1$$

$$NF3 = -(N_{wv}wv + N_{|v|v}|v|v|)/1^2$$

Again setting terms that include w, p, and q (heave, roll and pitch) equal to zero, and set number of known terms from Table II into section 1 of CSMP program (unknown coefficients set equal to zero).

TABLE II

Hydrodynamic Coefficients of Ship "D" for Nonlinear Terms (non-dimensional)

$$X_{rr} = 0.00005$$

$$X_{vr} = 0.00241$$

$$X_{vv} = -0.00341$$

$$Y_{v|v|} = -0.0416$$

$$N_{v|v|} = -0.01002$$

PROPULSION RATIO $\Delta \eta$

	$\eta \geq 0.45$	$-1.0 \leq \eta < 0.45$	$\eta \leq -1.0$
A1	-0.00004	-0.00032	-0.00117
A2	-0.00035	0.00070	-0.00100
A3	0.00099	0	-0.00085

Fig. 8 - Fig. 12 are the same results as Fig. 3 - Fig. 7. The results of computer program 2 are more accurate than the computer program 1, when compared with free running model test of NSRDC [Ref. 6]. Fig. 13 and Fig. 14 when studying the stability of the ship by applying a force moment to the ship in computer program, set

NF1 0.0001*(STEP(10.0)-STEP(10.01)) (can use NF1 be-
1 ause NF1 in this program equal zero)

Study direction stability of the ship by plotting direction of the ship (used advance VS. transfer ship length) and check heading angle of the ship by plotting YAW VS. TIME.

[illegible]

```

PARAM YVCT=-C.CC25
PARAM YV=-0.0063
PARAM LC=1.0
PARAM YXDCI=-C.CC02
PARAM YR=0.004
PARAM NVDCI=-C.CC01
PARAM NRDCI=-C.CC02
PARAM IZ=0.0003
PARAM NV=-0.0012
PARAM NR=-0.0012
PARAM XL=-0.0012
PARAM KA2=0.
PARAM KA3=0.
PARAM KB2=0.
PARAM KB3=0.
PARAM KC1=0.
PARAM KC2=0.
PARAM KC3=0.
PARAM KL2=0.
PARAM KL3=0.
PARAM KE1=0.0
PARAM KE2=0.
PARAM KE3=0.
PARAM XKK=C.CC005
PARAM XVR=0.00241
PARAM XVV=-0.CC341
PARAM AL=-0.00032
PARAM AZ=C.CC007
PARAM A3=C.0
PARAM YIVIV=-0.0416
PARAM NIVIV=-C.01C02
PARAM XCC=0.
PARAM XRF=0.
PARAM XAC=0.
PARAM YPC=0.
PARAM YVVI=0.
PARAM YVVI1=0.
PARAM YVVC=0.
PARAM NVC=0.
PARAM NVP=0.
PARAM NIVIR=0.
PARAM NV=0.
PARAM NC=C.
*SECTION 2 -PARAMETERS CALCULATIONS
INITIAL

```

2A - ALL PARAMETERS MUST BE DEFINED AND IN SEQUENCE:
 *AAC, AAL, AAE, AAF, ABA, ABB, ACC, ABD, ABE, ABF, ACA, ACB, ACC, ACC, ACE
 *ACF, ACA, ADB, ADD, ADE, ACF, AEA, AEB, AEC, AED, AEE, AEF, AFA, AFB, AFC, AFE, AFF

[illegible]

```

NF=NF1+NF2+NF3
NA1=-LC*(XVC**2+XRR**R**2+XRP**R**P)
NA2=-IML#V**R+XVR#V**C**W**C-ML**W**C)
NA3=-XVV**V**2)/LC
NA4=-A1#U**2+A2#U*UC+A3*UC**2)/LC
NB1=-LC#YPC**P**Q
NB3=-YWP**W**V+YVIR1#V*ABR-ML#U**R+ML**W**P)
NB5=-YPC**P**Q+(Y-IY-IY)*P**Q
NF2=-INWP**W**P+NLVIR#ABV**R)/LC
NF3=-INWV**W**V+NLVIR#ABV**V)/LC**2
IF1=KAI#D1+KA2#D2+KA3#D3+NA
IF2=KAI#D1+KB2#D2+KC3#D3+NC
IF3=KAI#D1+KC2#D2+KE3#D3+NE
IF4=KAI#D1+KE2#D2+KF3#D3+NF
IF5=KAI#D1+KF2#D2+KF3#D3+NF
IF6=KAI#D1+KF2#D2+KF3#D3+NF
ACCT=(COFBA**I1+COFBA**I2+COFBA**I3+COFAD**I4+COFAE**I5+COFAF**I6)/DEL
BCDDCT=(COFBA**I1+COFBA**I2+COFBA**I3+COFBC**I4+COFBC**I5+COFBC**I6)/DEL
CCLLCT=(COFBA**I1+COFBA**I2+COFBA**I3+COFBC**I4+COFBC**I5+COFBC**I6)/DEL
EDDOT=(COFEA**I1+COFEA**I2+COFEA**I3+COFEC**I4+COFEC**I5+COFEC**I6)/DEL
ACCT=INTGR(.,.,CCCT)
BCDDCT=INTGR(.,.,CCCT)
CCLLCT=INTGR(.,.,CCCT)
EDDOT=INTGR(.,.,CCCT)
A=INTGR(.,.,CCCT)
B=INTGR(.,.,CCCT)
C=INTGR(.,.,CCCT)
D=INTGR(.,.,CCCT)
E=INTGR(.,.,CCCT)
F=INTGR(.,.,CCCT)
*SECTION 5-OUTPL CHARACTERISTICS
TIMER PREPARE YH,XH
FINTIM=240.0,OUTDEL=1.0,PRODEL=1.0
END
PARAM CR=-0.4365
ENC
PARAM CR=-0.6111
ENC
STCF

```

```

AAA=XUDCT-ML
AA6=0.
AAC=0.
AAC=0.
AAE=0.
AAF=0.
A2A=0.
2BB=YVDDT-ML
ABC=0.
ABD=0.0
ABE=0.
ABF=NVDDT/LC
ACA=0.
ACB=0.
ACC=1.0
ACD=0.
ACE=0.
ACF=0.0
ADA=0.0
ADB=0.0
ADC=1.0
ADE=0.
ADF=0.0
AEE=0.
AEE=1.0
AEE=0.
AFA=0.
AFB=LC*YRDC
AFC=0.
AFD=0.0
AFE=0.
AFF=NRDDT-IZ
SECTION 2A
DELE=VALUE(AAA,0,0)
CCFAA=VALUE(AAA,1,1)
CCFAB=VALUE(AAA,2,1)
CCFAC=VALUE(AAA,3,1)
CCFAD=VALUE(AAA,4,1)
CCFAE=VALUE(AAA,5,1)
CCFAF=VALUE(AAA,6,1)
CCFBA=VALUE(AAA,1,2)
CCFBB=VALUE(AAA,2,2)
CCFBC=VALUE(AAA,3,2)
CCFBD=VALUE(AAA,4,2)

```

*END

CYANIDE

*SECTION 2-DEFINITIONS

```

COMMON
      FUNCTION VALUE(Y,I,M)
      DIMENSION X(6,6),Y(6,6)
      DO 1 M1=1,6
      DO 1 M2=1,6
      1  X(M1,M2)=Y(M1,M2)
      IF(1.EC.O) GO TO 100
      X(1,1)=C.
      X(1,2)=C.
      X(1,3)=C.
      X(1,4)=C.
      X(1,5)=C.
      X(1,6)=C.
      X(2,M)=C.
      X(3,M)=C.
      X(4,M)=C.
      X(5,M)=C.
      X(6,M)=C.
      X(1,M)=1.
      CCNTINUE
      CCNTINUE(6,49,I,M)
      DETERMINANT FOR COF,11,11,':')
      49  FCRMAT(1,1,6)
      51  FCRMAT(10(1X,EL3.6))
      50  CCNTINUE
      N=6
      CC=1.D0
      DO 34 L=1,N
      KP=0
      Z=0.0
      11  IF(2-ABS(X(K,L)))11,12,12
      Z=ABS(X(K,L))
      KP=K
      CCNTINUE
      12  IF(L-KP)13,20,20
      13  DO 14 J=L,N
      Z=X(L,J)
      X(L,J)=X(KP,J)
      X(KP,J)=Z
      CC=-DD
      20  IF(L-N)131,40,40
      31  LP1=L+1
      DO 34 K=LP1,N
      IF(X(K,L))32,34,32
      32  RATIO=X(K,L)/X(L,L)

```



```

33      GO 33 J=LPI,N
34      X(K,J)=X(K,J)-RATIO*X(L,J)
35      CCNTINUE
36      CC 41 K=1,N
37      CC L=CD*X(K,<)
38      L=CD
39      VALUE=D
40      WRITE(6,52) I,M,VALUE
41      FORMAT(' ',COF,I1,I1,'=',E15.6)
42      RETURN
43      END
ENDJCB

```

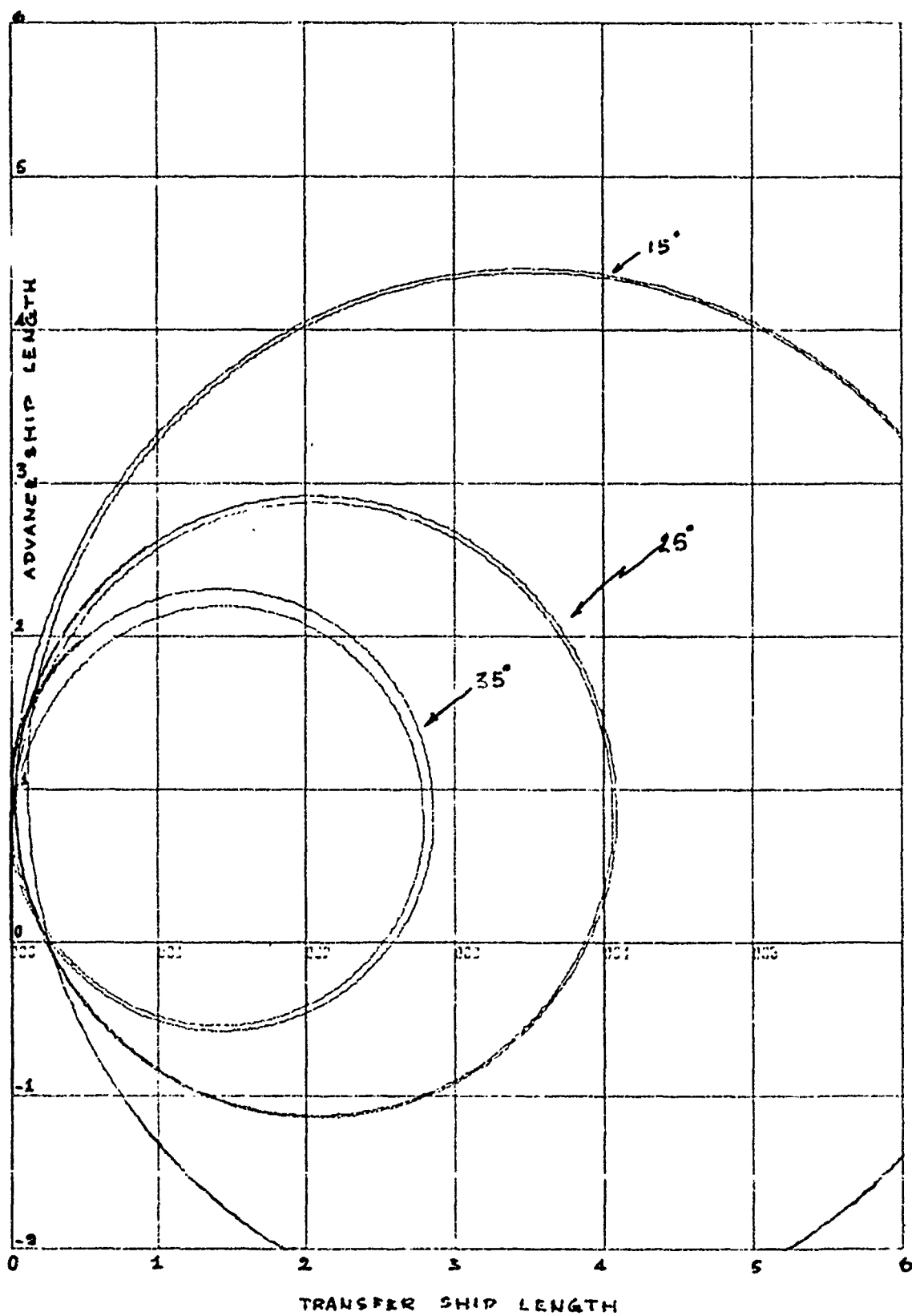


FIG. 8 ADVANCE VS. TRANSFER SHIP LENGTH (DR. 15°, 25° & 35°)

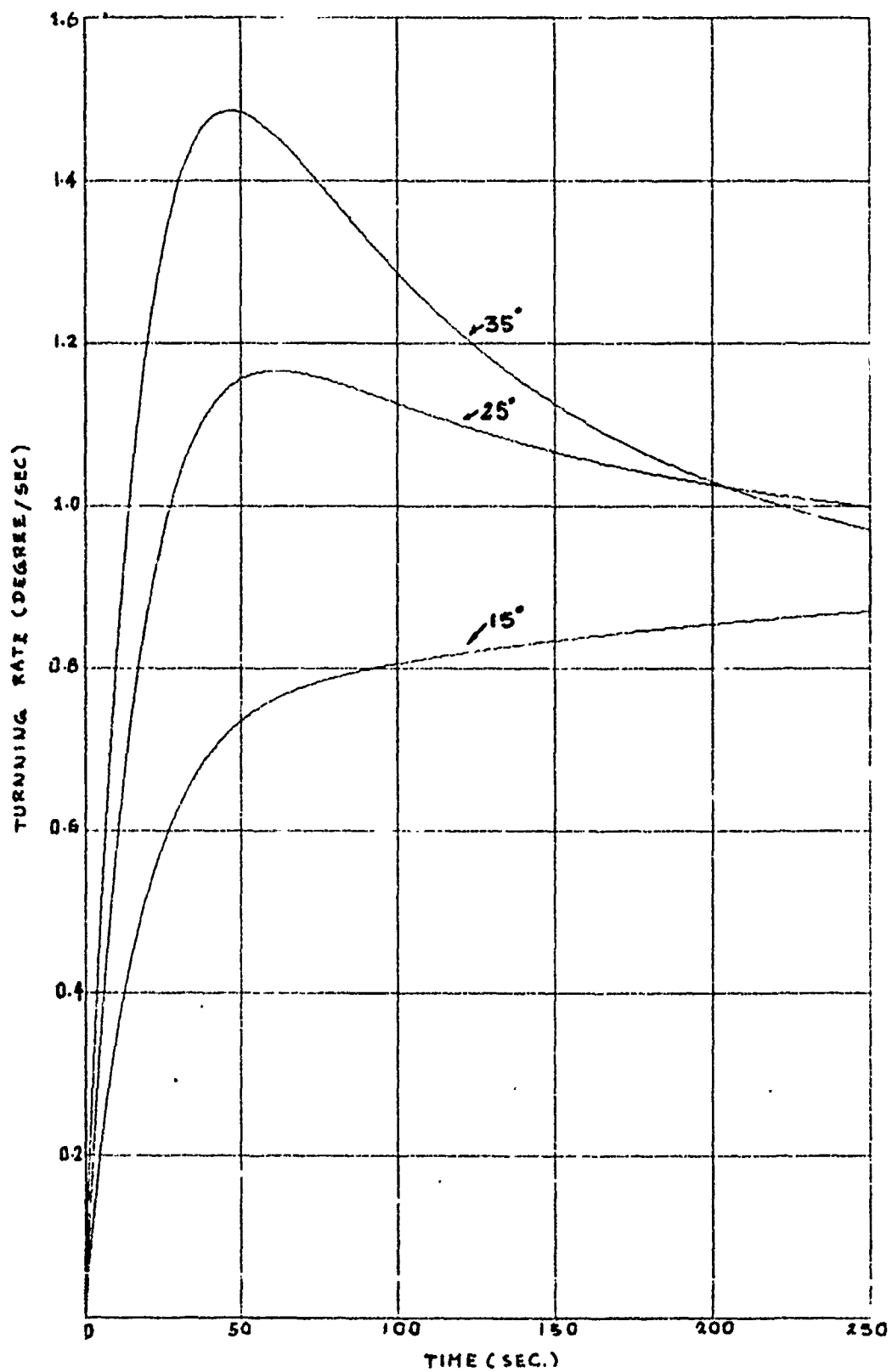


FIG. 9 TURNING RATE AS A FUNCTION OF TIME
(RUDDER 15°, 25 & 35°)

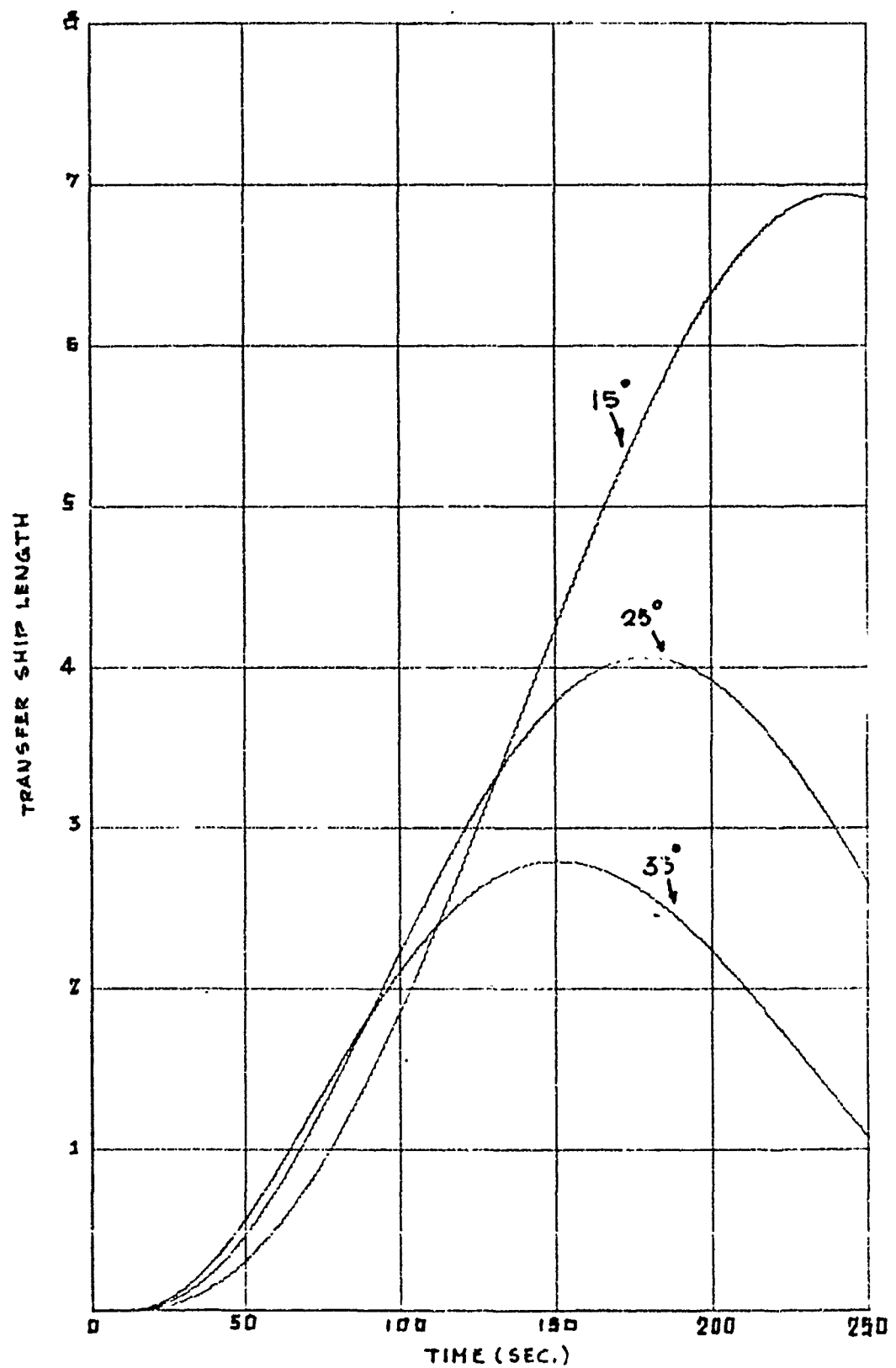


FIG. 10 TRANSFER SHIP LENGTH VS. TIME
(RUDDER 15°, 25° & 35°)

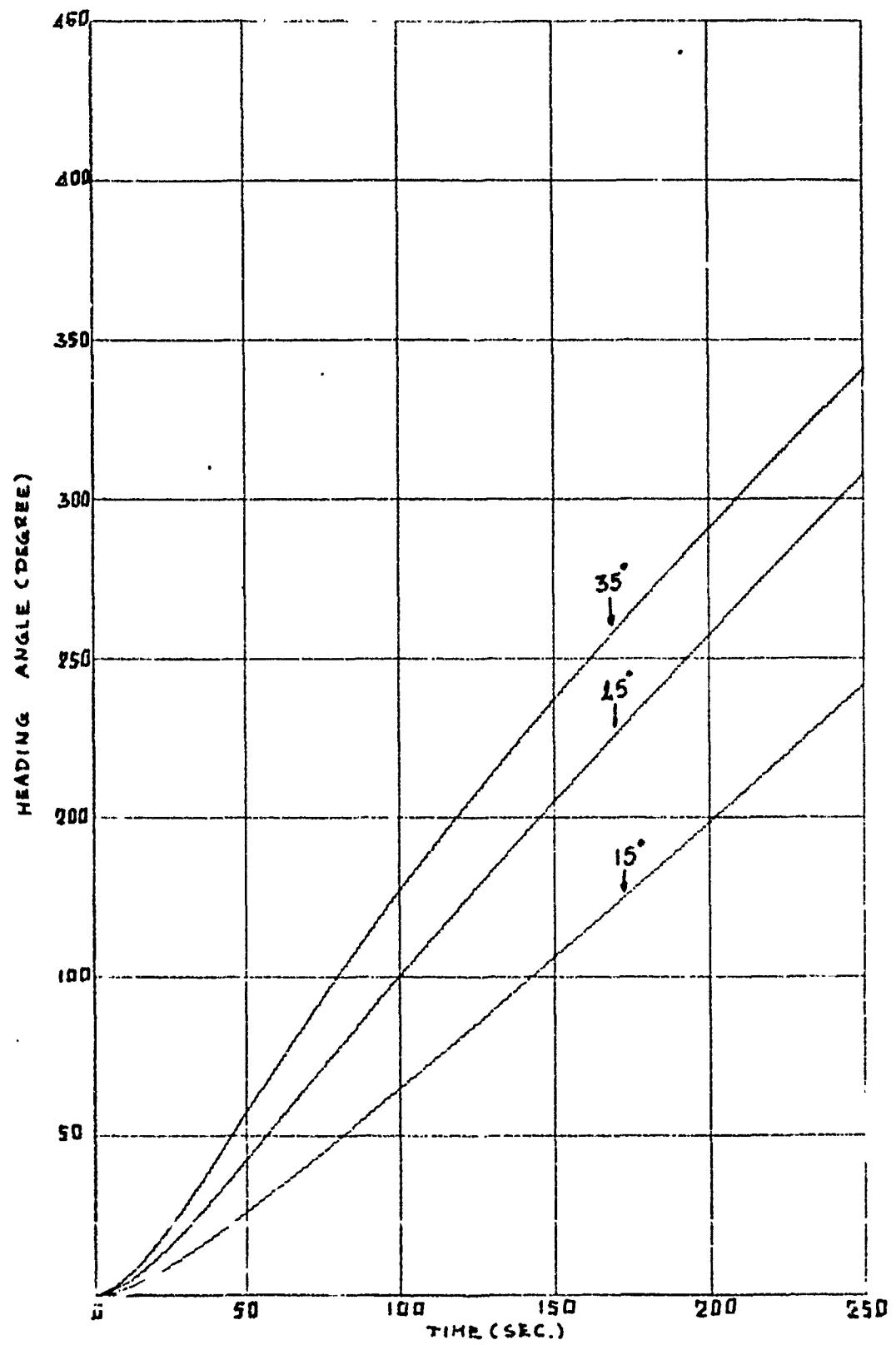


FIG. II HEADING ANGLE AS A FUNCTION OF TIME
[RUDDER 15°, 25° & 35°]

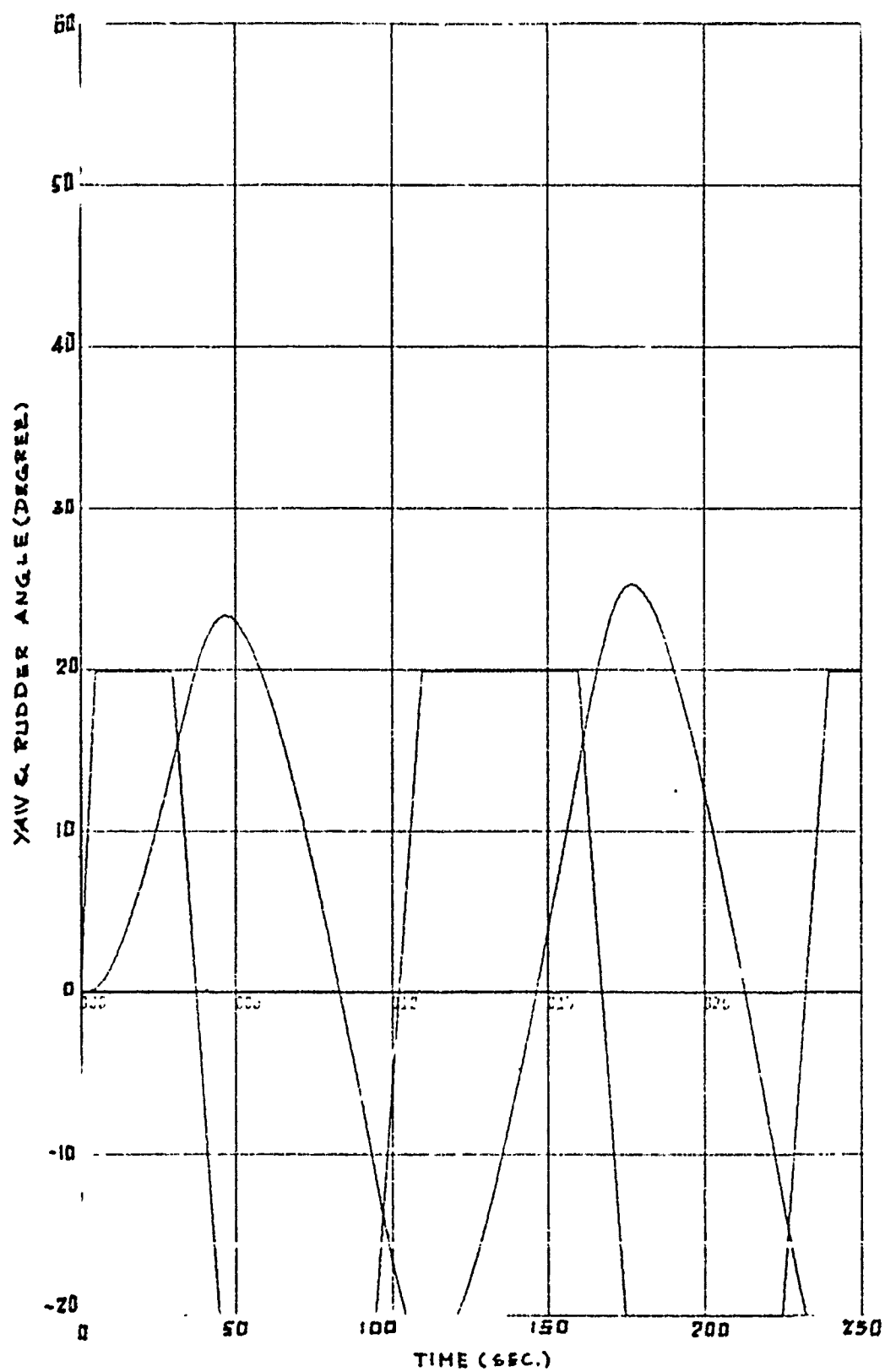


FIG. 12 YAW & RUDDER ANGLE VS. TIME
(ZIG-ZAG MANOEUVRE)

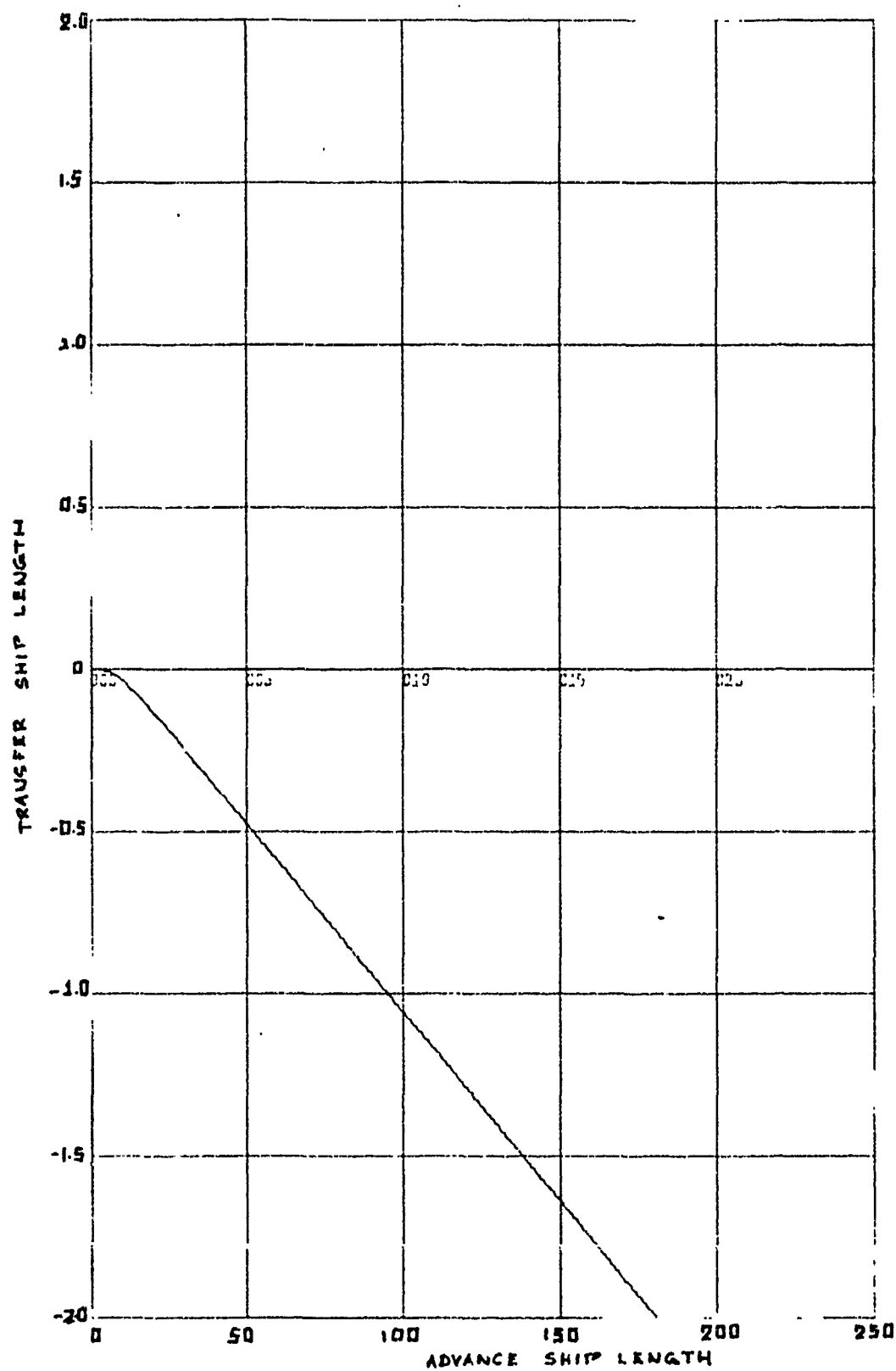


FIG.13 DIRECTION OF THE SHIP WHEN EXTERNAL MOMENT
FORCE APPLIED TO THE SHIP

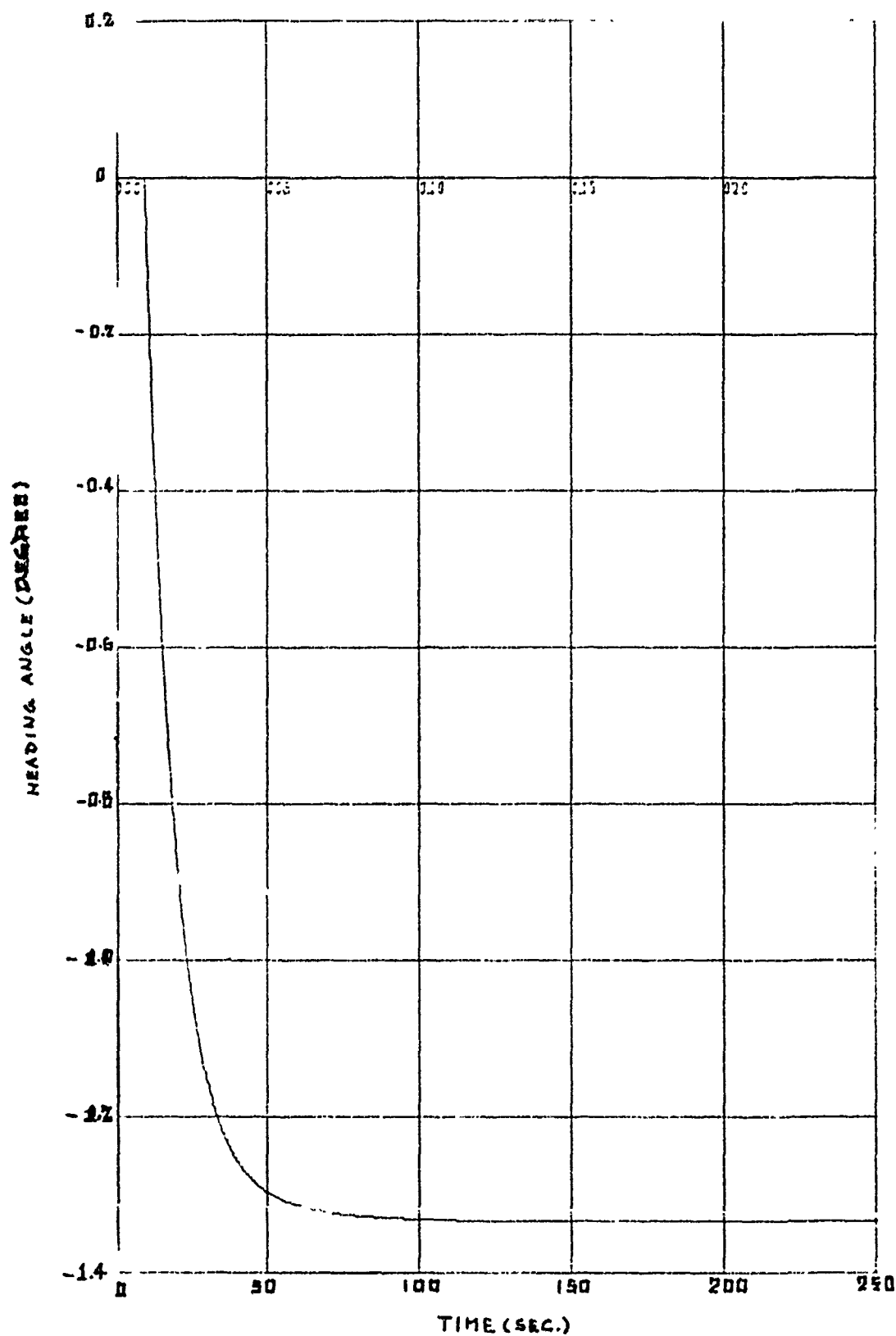


FIG. 14 HEADING ANGLE VS. TIME
(APPLIED FORCING MOMENT TO THE SHIP)

IV. CONCLUSIONS

The equation of motion of surface ship and computer program developed here including all of six degrees of freedom, but the study in III concerns only three degrees of freedom (surge, sway, yaw) because hydrodynamic coefficients are not available; when the state of the art reaches the stage in which hydrodynamics coefficients are available, this computer program can be used in all six degrees of freedom.

Some results from III are not too perfect because the lack of some constants and coefficients such as the value of mass (m), initial velocity (ADOTO), command speed (UC), etc. But for study can adjust from curve for the model test [Ref. 6].

This computer program did not include some external effects such as effects of wave and wind, but these effects could be included in the program by adding terms to the IF equation.

The following implementations are suggested for the future work.

- A. Study all six degrees of freedom.
- B. Study for the effects of waves and wind.
- C. Study for control of the velocity and direction of the ship (by use of "MACROS", "PROCEDURE" or subprogram in CSMP).

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